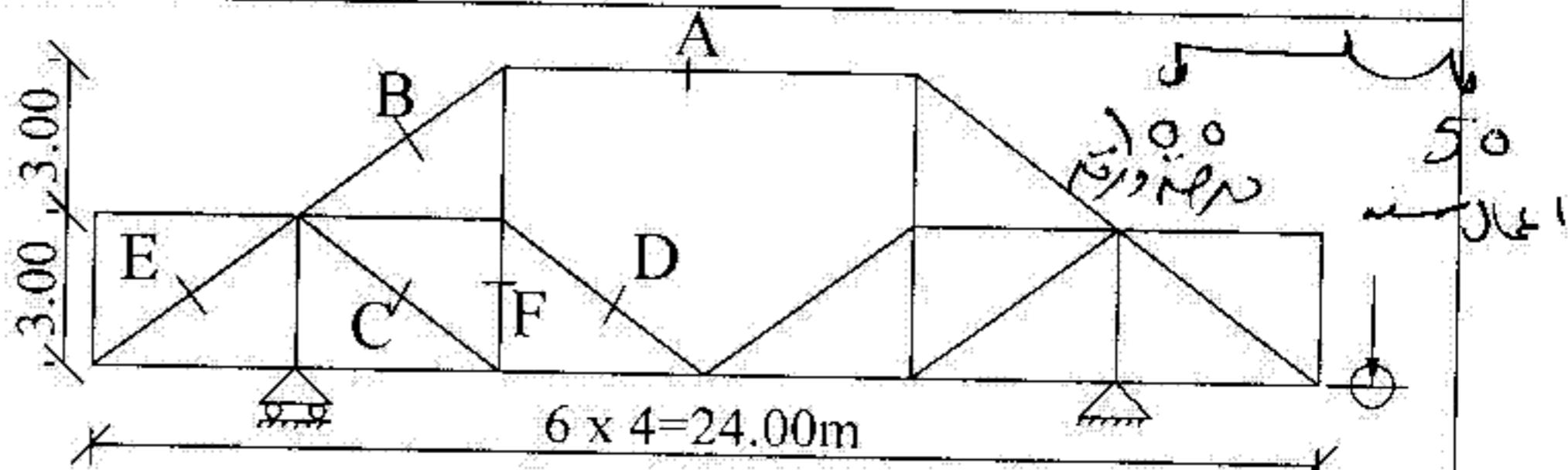


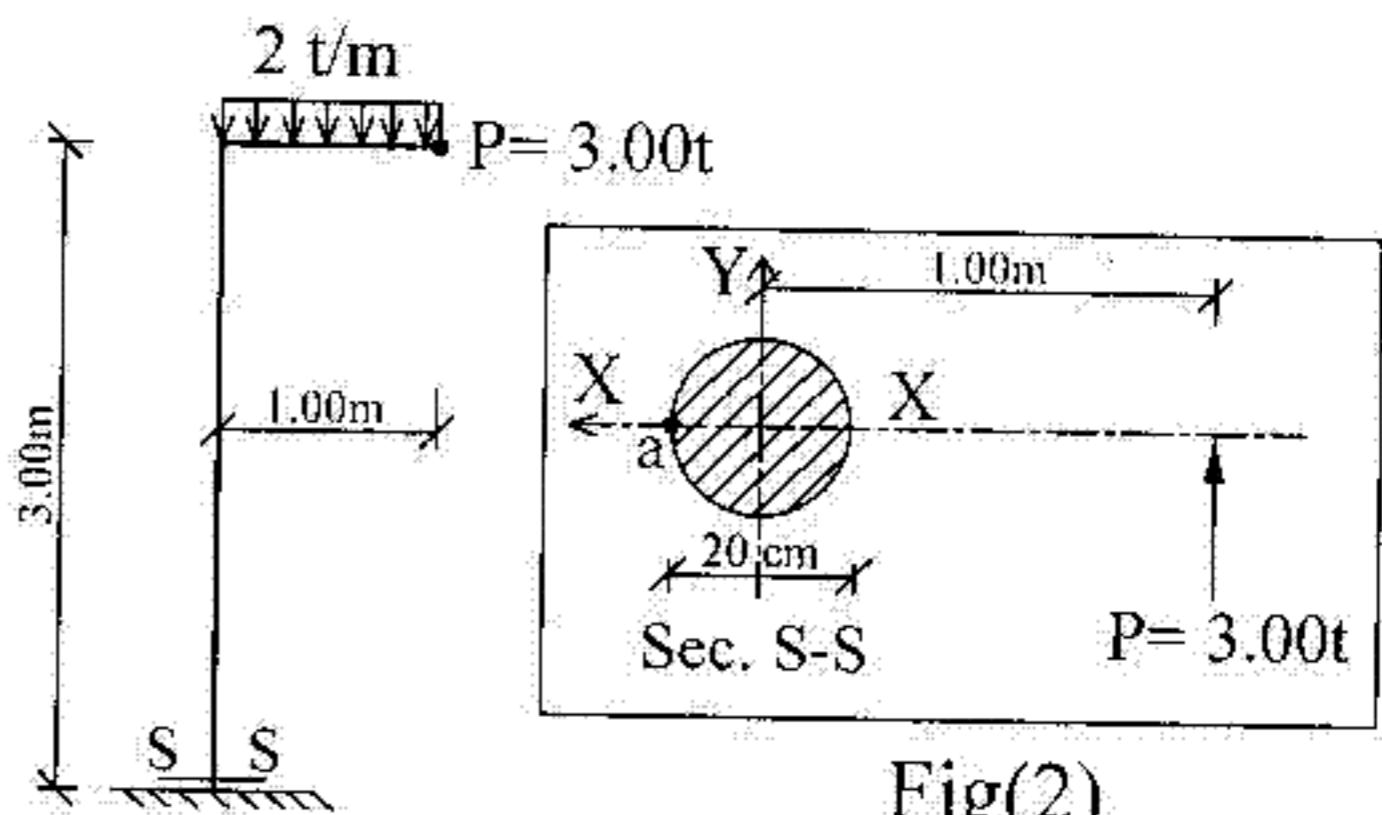
Question (1) (20%)

For the truss shown in Fig (1)
draw influence lines for the
marked members A, B,C,D,E,F



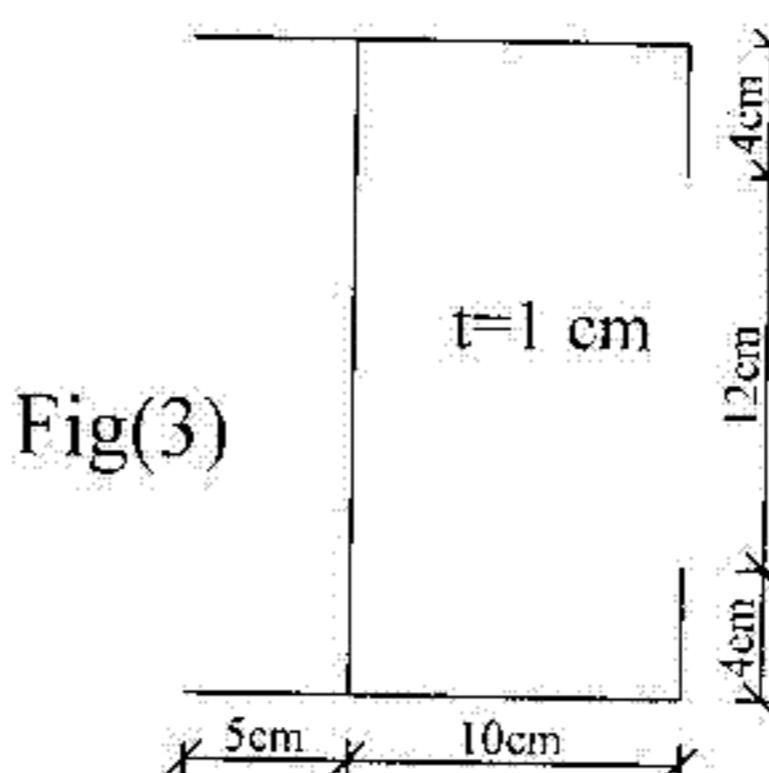
Question (2) (30%)

The shown steel column carries a uniform load
2 t/m and an out-of plane lateral load P of 3t.
Determine analytically and check graphically
the directions and stresses on the principal normal
and principal shear planes at point (a) of section
S-S shown in Fig. (2).



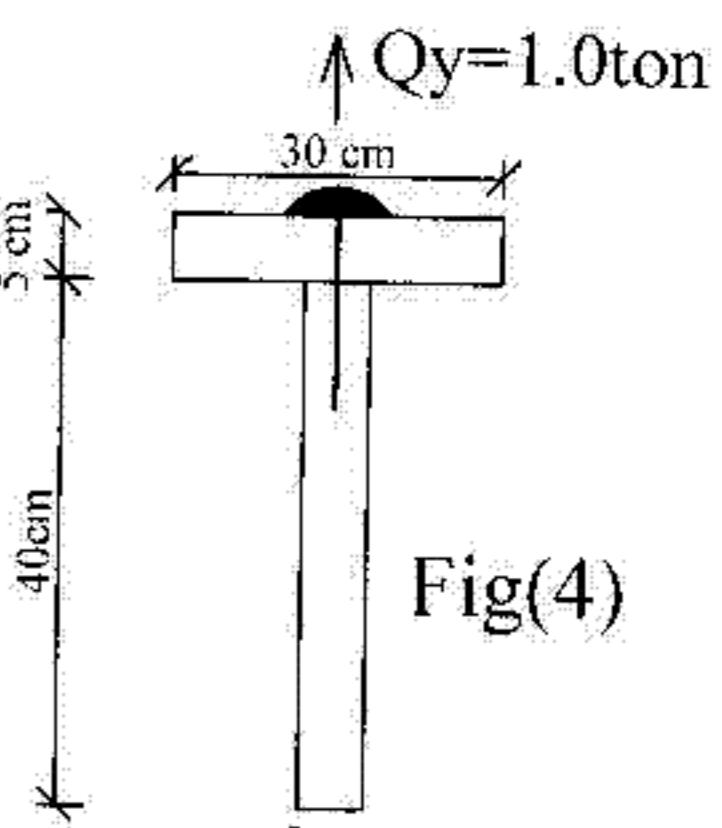
Question (3) (20%)

Find the shear center of the
section shown in Fig. (3) by
using the shear flow diagram



Question (4) (20%)

Nails are used to connect the two wooden planks to make them
act as one beam as shown in Fig. (4). Calculate the spacing (p) of
the nails if $Q_y=1.00$ ton and the allowable shearing force per nail
is 50 Kg.

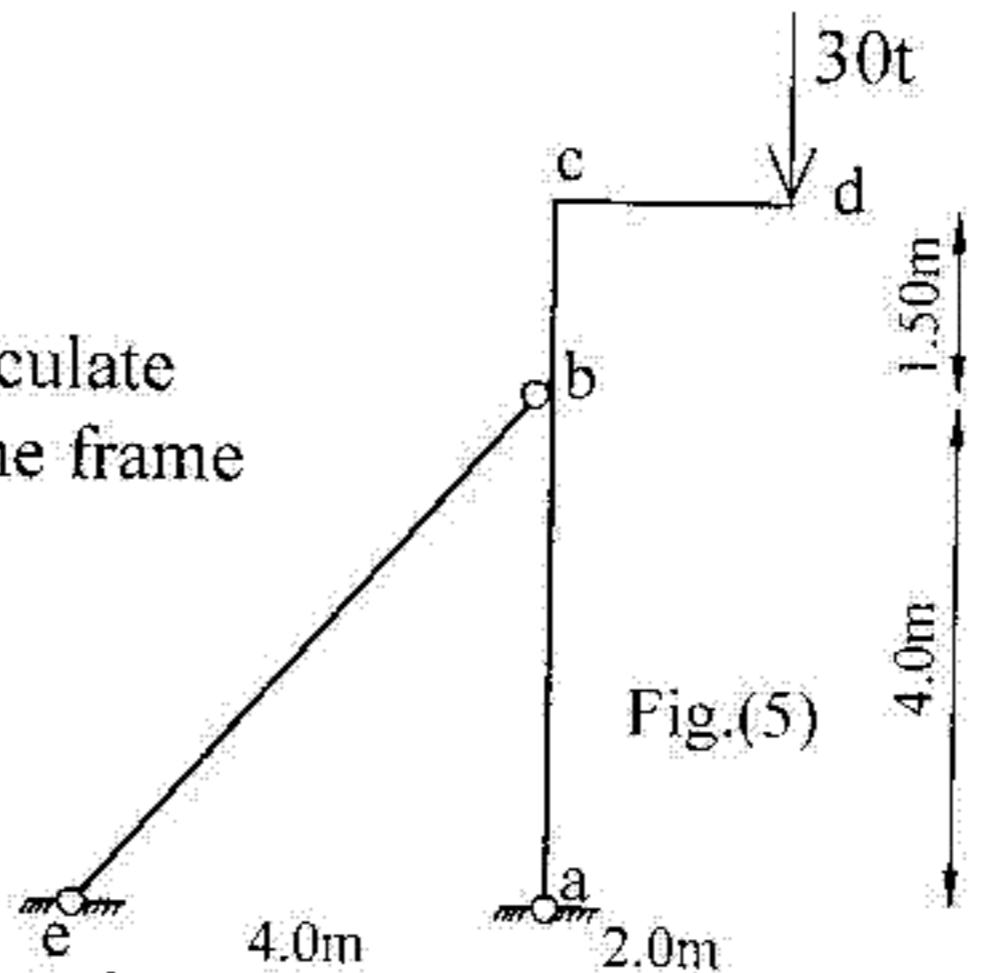


Question (5) (20%)

Using the method of virtual work, calculate
the vertical deflection of point d for the frame
shown in Fig. (5)

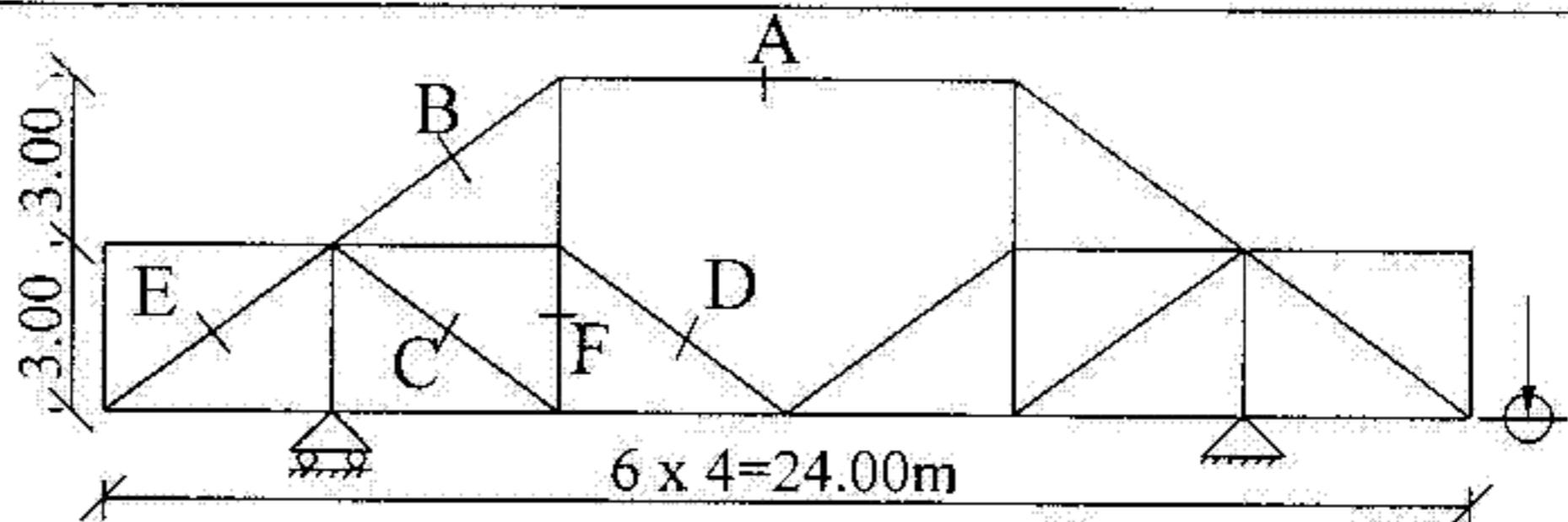
$$EI(\text{part abcd})=50,000 \text{ m}^2\text{t}$$

$$EA(\text{tie})=40,000 \text{ t}$$



Question (1) (20%)

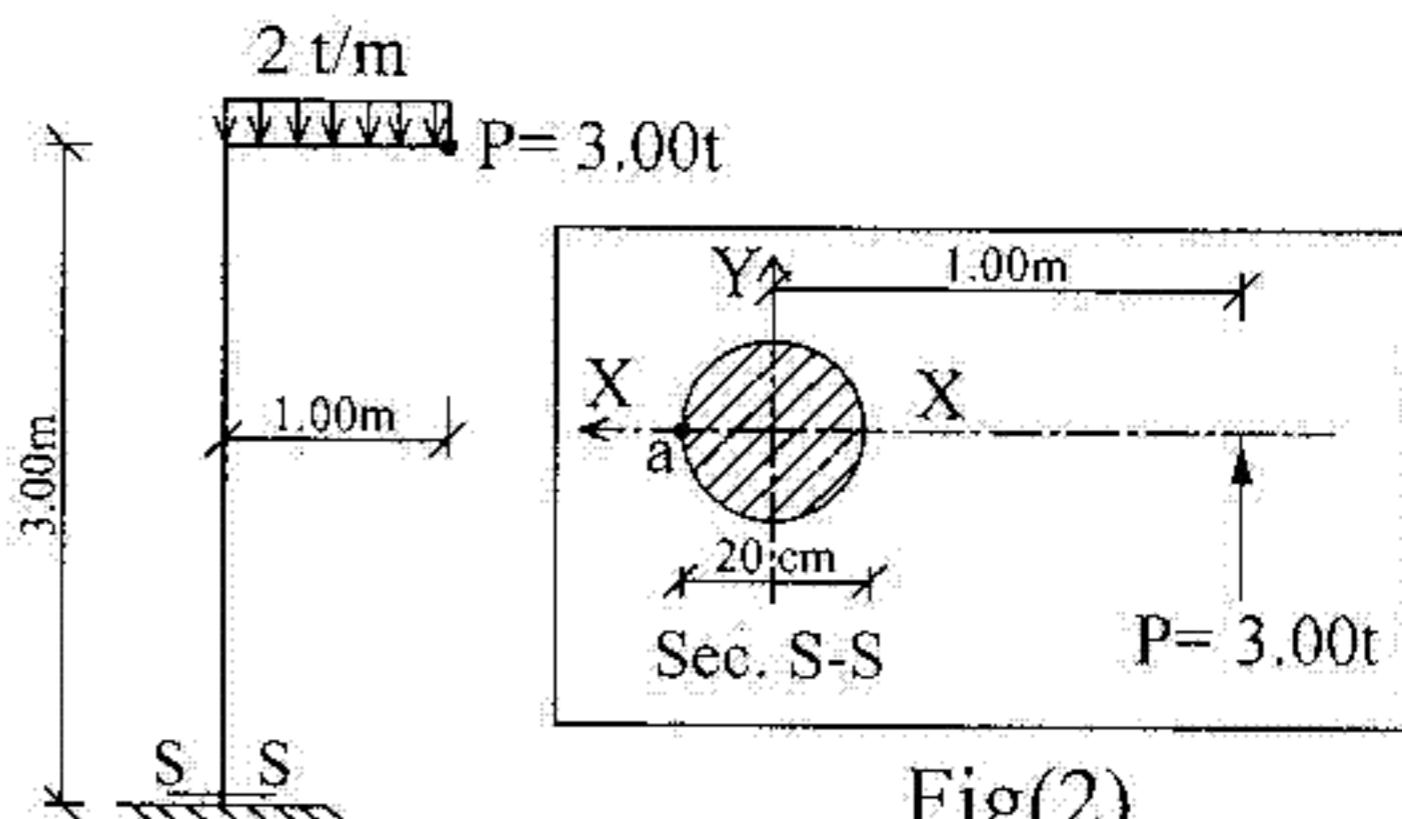
For the truss shown in Fig (1) draw influence lines for the marked members A, B,C,D,E,F



Fig(1)

Question (2) (30%)

The shown steel column carries a uniform load 2 t/m and an out-of plane lateral load P of 3t. Determine analytically and check graphically the directions and stresses on the principal normal and principal shear planes at point (a) of section S-S shown in Fig. (2).

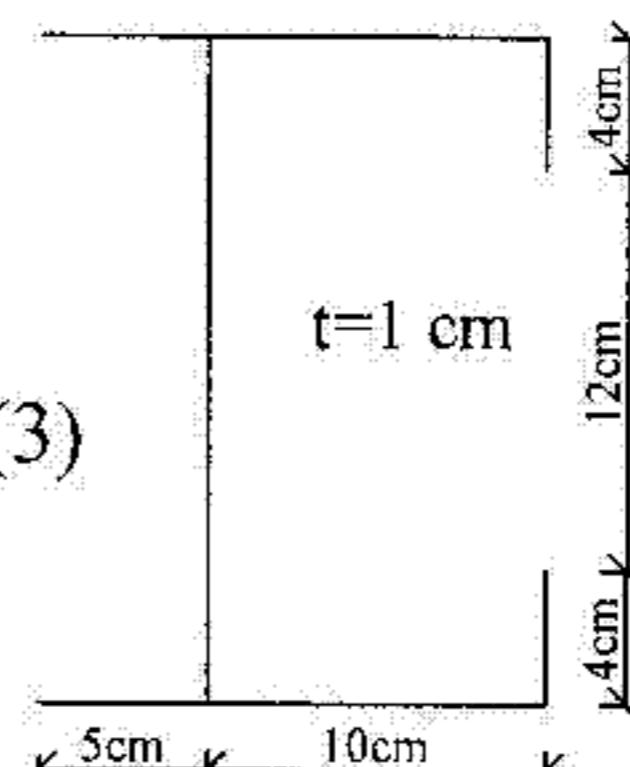


Fig(2)

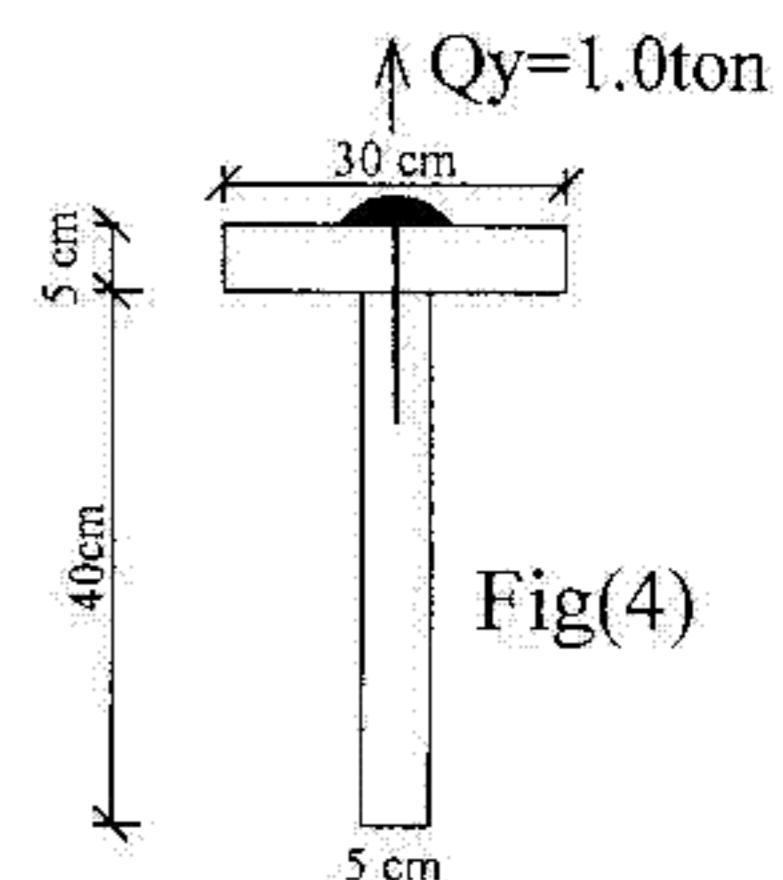
Question (3) (20%)

Find the shear center of the section shown in Fig. (3) by using the shear flow diagram

Fig(3)

Question (4) (20%)

Nails are used to connect the two wooden planks to make them act as one beam as shown in Fig. (4). Calculate the spacing (p) of the nails if $Q_y = 1.00$ ton and the allowable shearing force per nail is 50 Kg.



Fig(4)

Question (5) (20%)

Find the minimum permissible diameter d for the circular shaft shown in Fig. (5) if the allowable shearing stress = 500 kg/cm².

Hence, find the maximum angle of twist.

$$G = 500 \text{ t/cm}^2$$

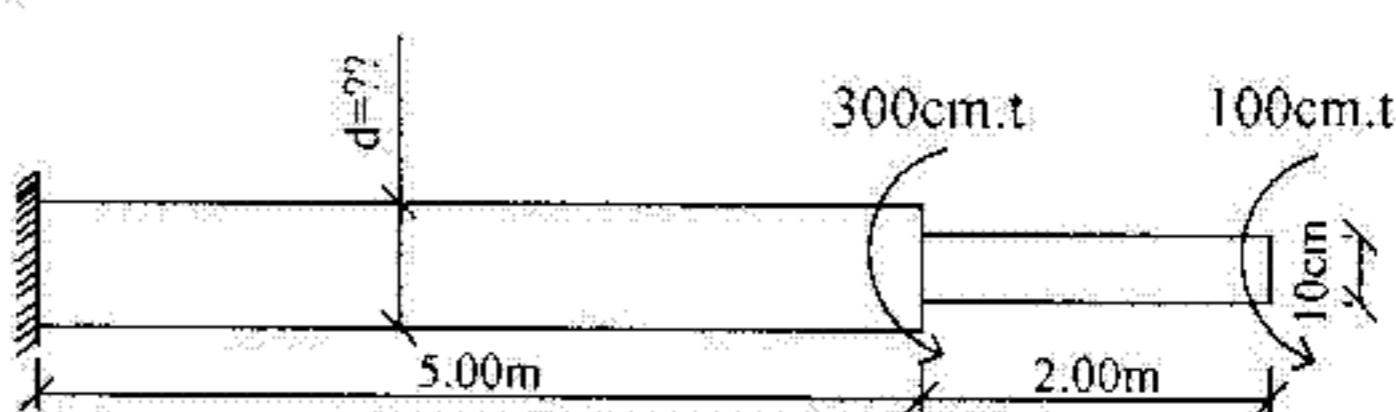
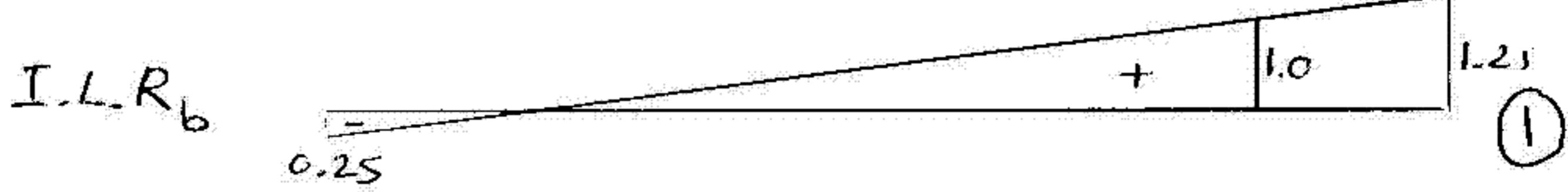
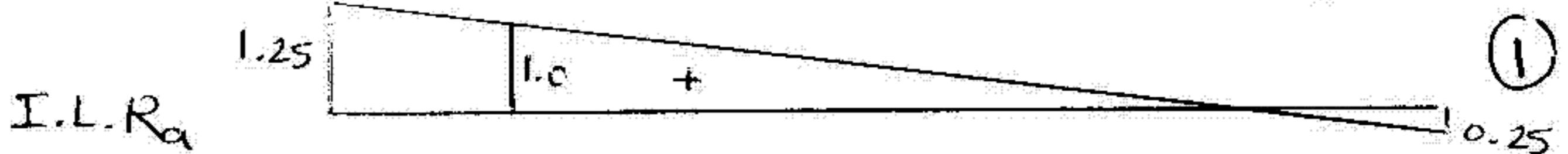
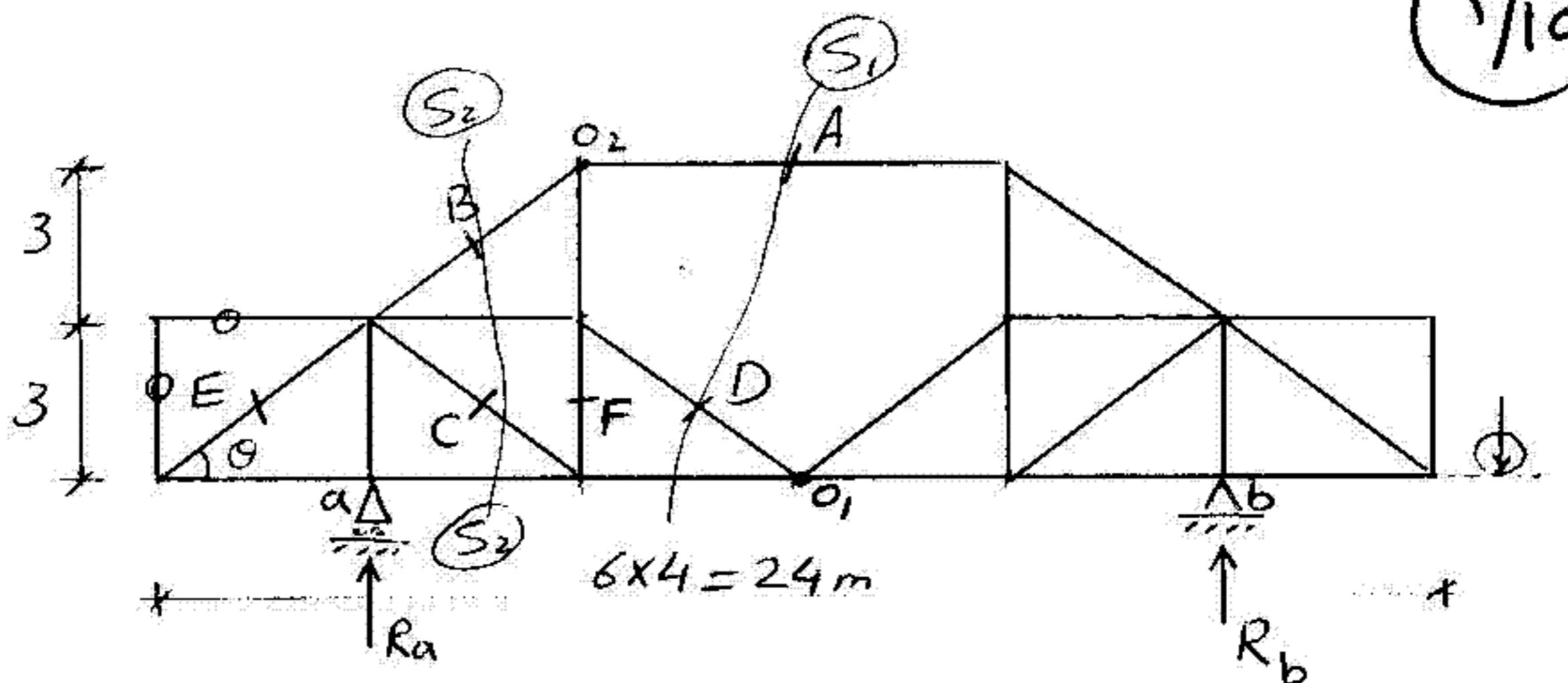


Fig.(5)

I
20

1/10

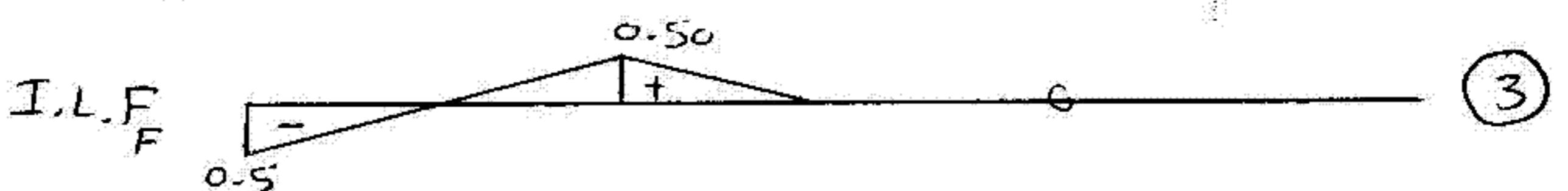
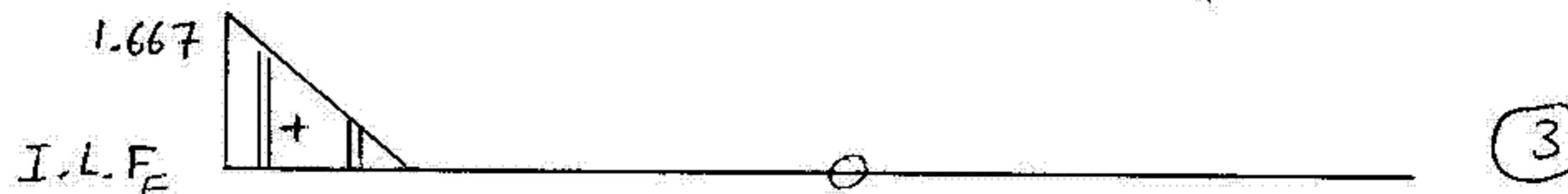
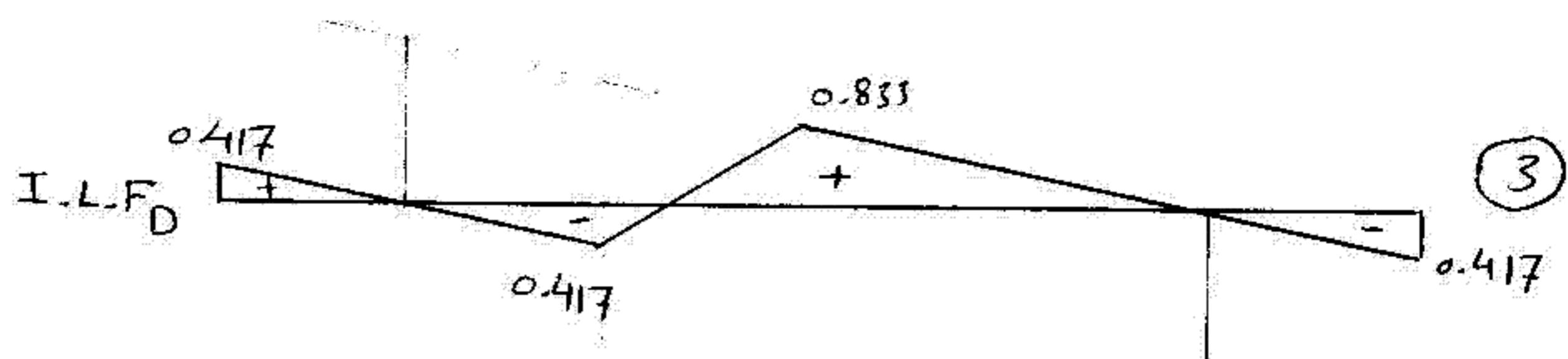
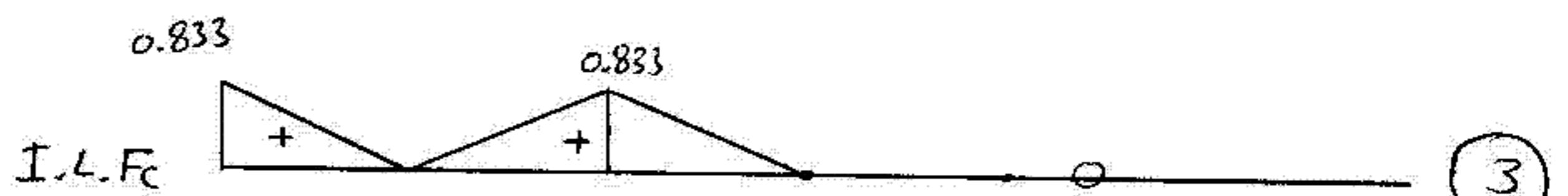
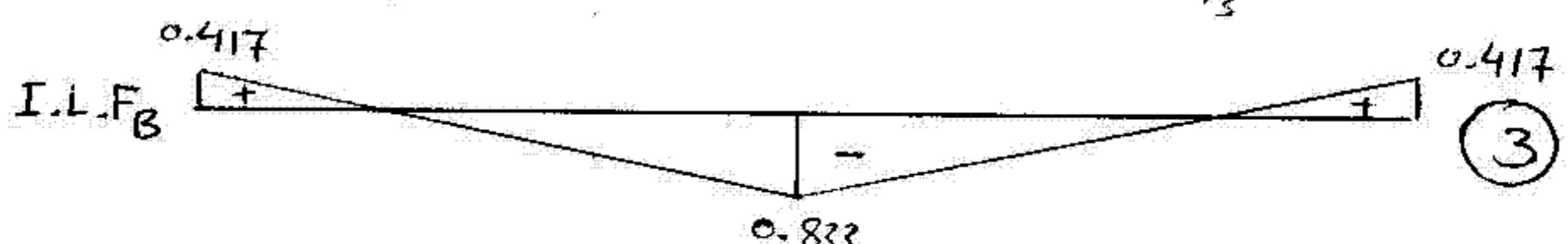
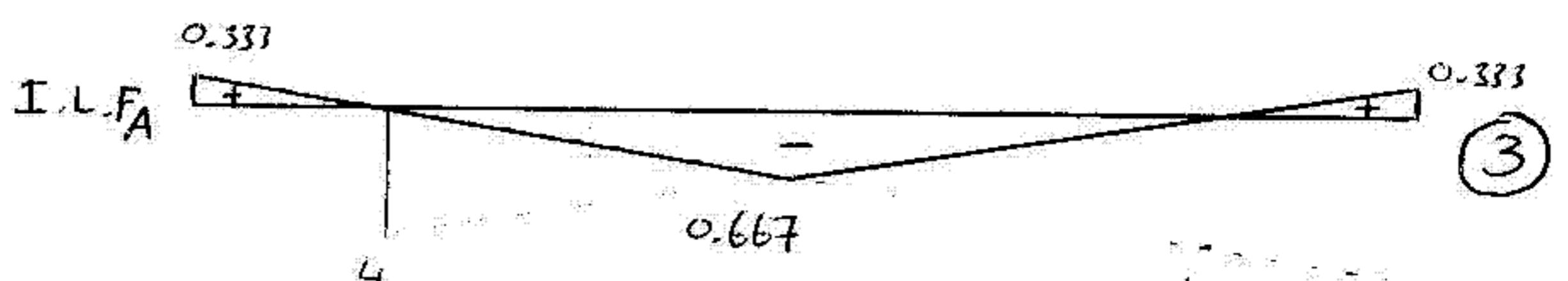


I.L. R_a , I.L. R_b

I.L. F_A

$\text{Sec}(S_1, -S_1)$

$\sum M_{O_1} = 0$



2

2/10

$$N = -2 \times 1 = -2 t$$

$$Q_x = 0$$

$$Q_y = 3t$$

$$M_x = -3 \times 3 = -9 \text{ t.m.}$$

$$M_y = +2 \times 1 \times 0.5 = +1.0 \text{ t.m.}$$

$$M_{tx} = 3 \times 1 = 3 \text{ t.m.}$$

at point a ($x = -10 \text{ cm}$, $y = 0$)

$$\sigma = \frac{N}{A} + \frac{M_x}{I_x} y + \frac{M_y}{I_y} x$$

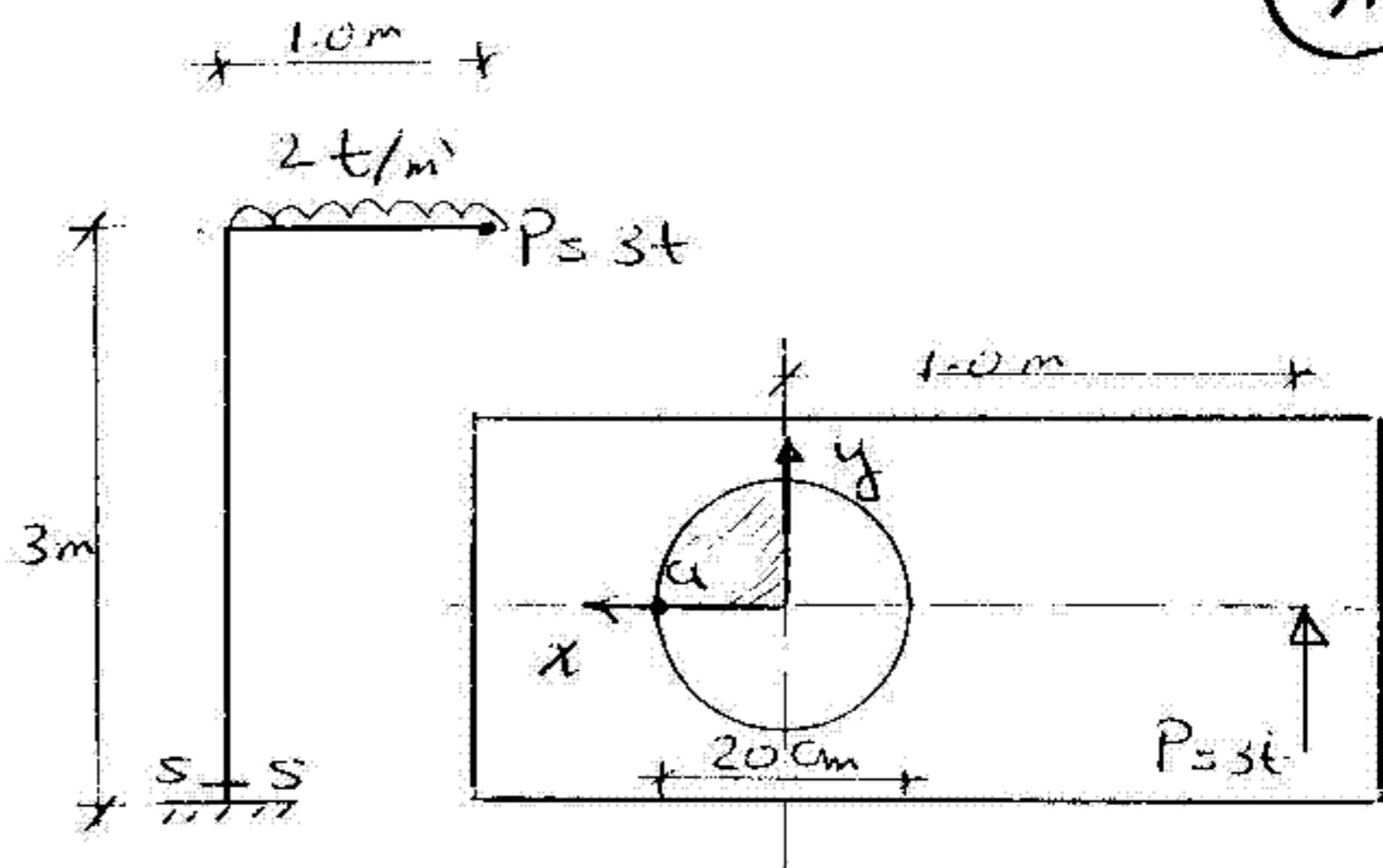
$$A = \pi(10)^2 = 314.16 \text{ cm}^2$$

$$I_x = I_y = \frac{\pi(10)^4}{4} = 7853.98 \text{ cm}^4$$

$$\sigma_a = \frac{-2}{314.16} + \frac{M_x}{I_x} y + \frac{1.0 \times 100}{7853.98} \times (+10)$$

$$\sigma_a = 0.121 \text{ t/cm}^2$$

$$\sigma = 121 \text{ kg/cm}^2$$

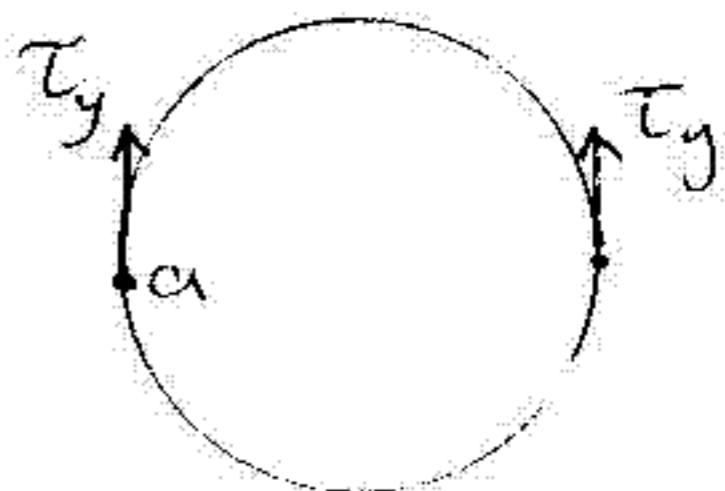


3/10

due to Shear

$$\tau_y = \frac{Q_y S_x}{I_x b} = \frac{3 * \left[\frac{\pi(10)^2}{2} * \frac{4(10)}{3\pi} \right]}{7853.98 * 20} = 0.01273 \text{ t/cm}^2$$

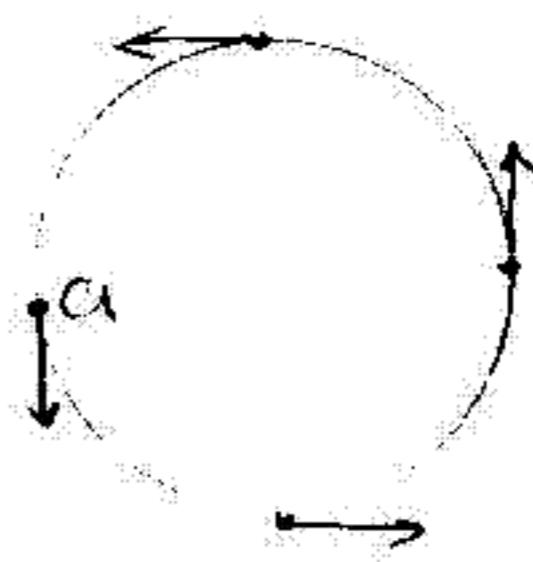
$$\tau_y = 12.73 \text{ kg/cm}^2$$

due to torsion

$$\tau_t = \frac{M_t * R}{I_p} = \frac{3 * 10 * 10}{\frac{\pi(10)^4}{2}} = 0.19098 \text{ t/cm}^2$$

$$\tau_t = 190.98 \text{ kg/cm}^2$$

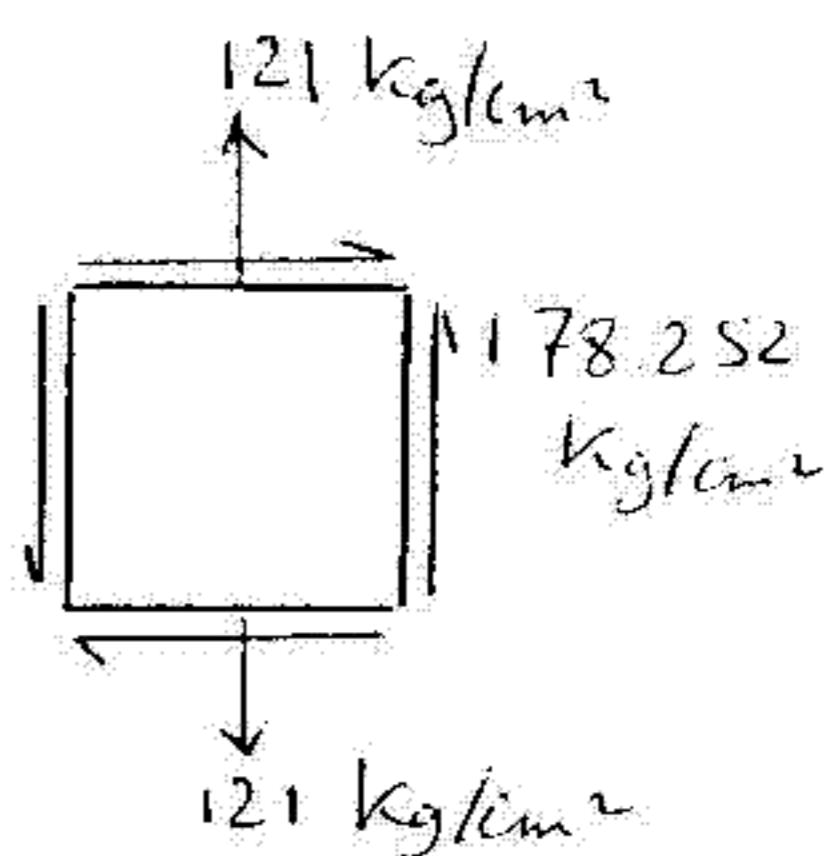
$$\tau_{\text{total}} = 12.73 - 190.98 = -178.25 \text{ (a) kg/cm}^2$$

at point a

$$\alpha_x = 0$$

$$\alpha_y = +121 \text{ kg/cm}^2$$

$$\tau_x = -\tau_y = 178.252 \text{ kg/cm}^2$$



(4/10)

Analytical Method

$$\begin{aligned}\sigma_{\max} &= \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau^2} \\ &= \left(\frac{0+121}{2}\right) \pm \sqrt{\left(\frac{0-121}{2}\right)^2 + (178.252)^2}\end{aligned}$$

$$\sigma_{\max} = 248.74 \text{ kg/cm}^2$$

$$\sigma_{\min} = -127.74 \text{ kg/cm}^2$$

$$\tan 2\alpha = \frac{-\tau_x}{\left(\frac{\sigma_x - \sigma_y}{2}\right)} = \frac{-178.252}{\left(\frac{0-121}{2}\right)}$$

$$2\alpha = 180^\circ + 71.252$$

$$\alpha = 125.63^\circ$$

$$\begin{aligned}\tau_{\max} &= \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_x^2} \\ &= \pm \sqrt{\left(\frac{0-121}{2}\right)^2 + (178.252)^2}\end{aligned}$$

$$\tau_{\max} = +188.24 \text{ kg/cm}^2$$

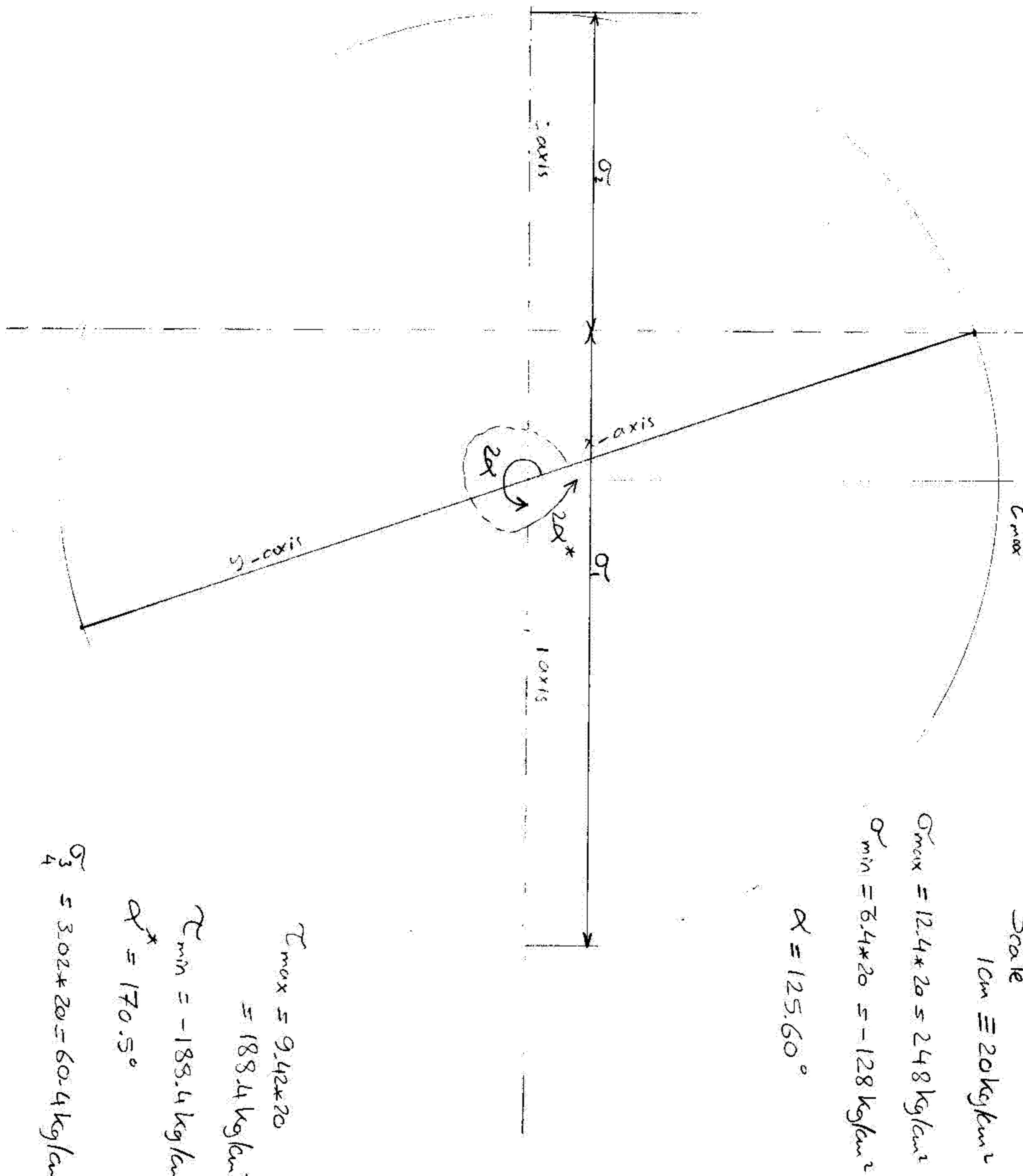
$$\tau_{\min} = -188.24 \text{ kg/cm}^2$$

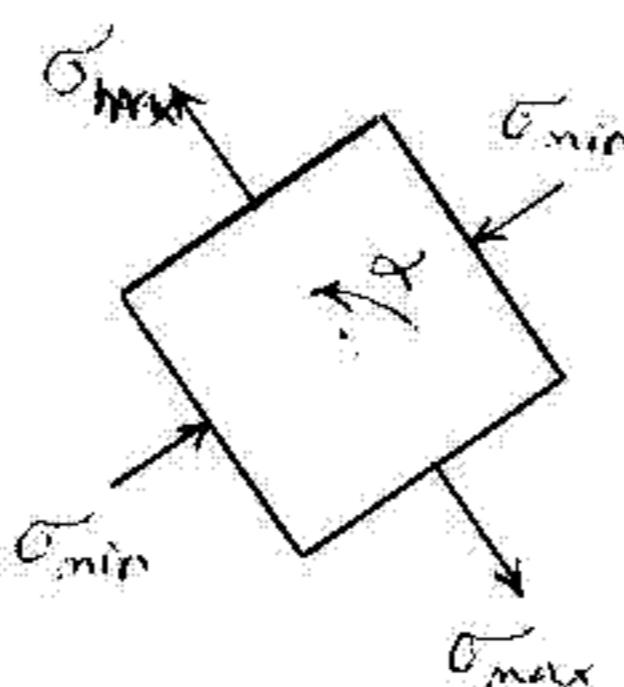
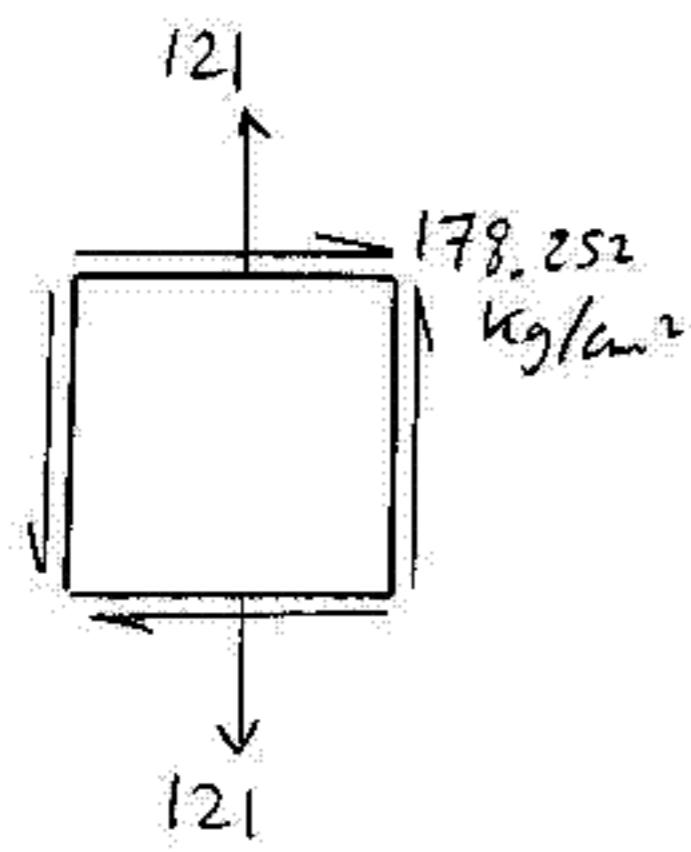
$$\alpha^* = \alpha + 45^\circ = 170.63^\circ$$

$$\sigma_3 = \frac{\sigma_x + \sigma_y}{2} = \frac{0+121}{2} = 60.5 \text{ kg/cm}^2$$

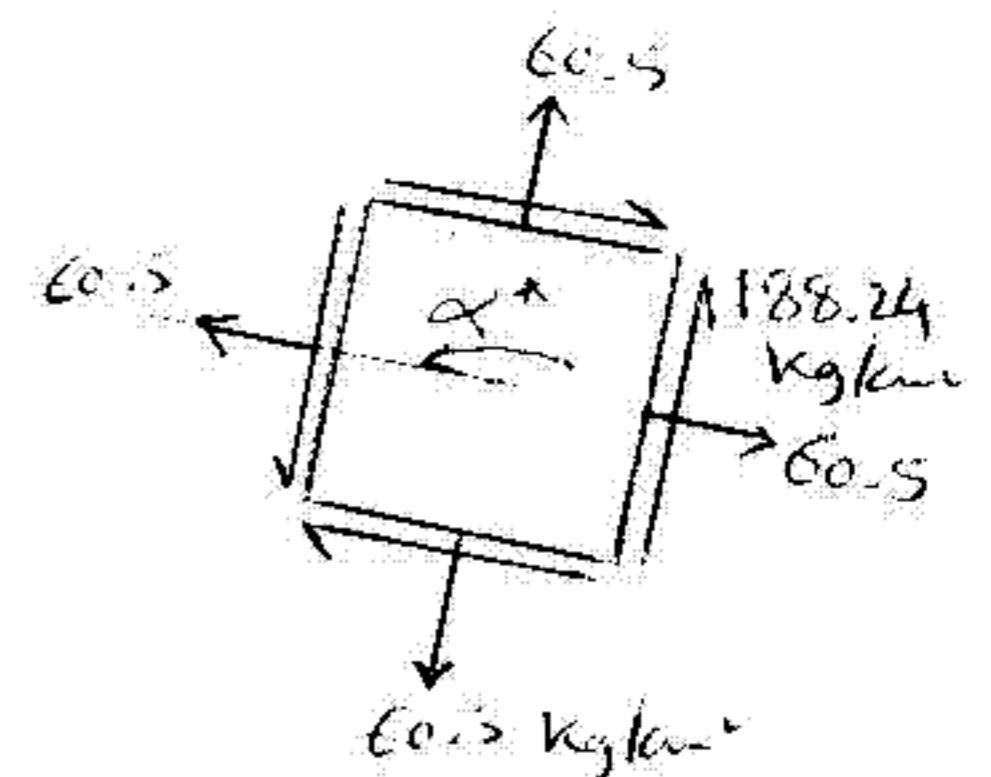
Graphical Method

5/10





Principal Normal
Stresses



Principal Shear
Stresses

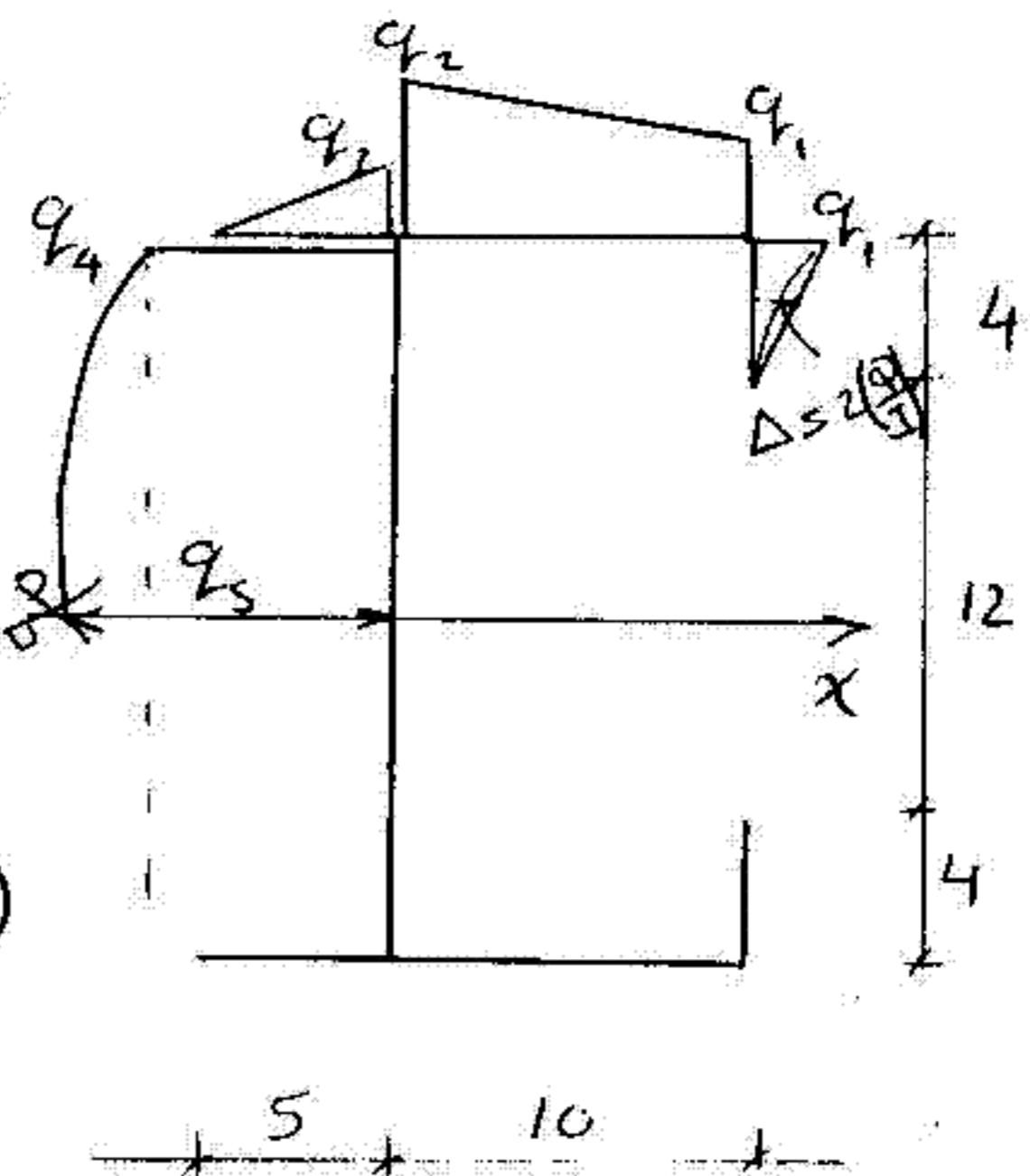
3

20

7/10

$$q = \frac{\Phi_y}{I_x} S_x$$

$$I_x = \left[\begin{array}{l} \frac{10(1)^3}{12} + 10(1)(10)^2 \\ + \frac{5(1)^3}{12} + 5(1)(10)^2 \\ + \frac{1(4)^3}{12} + 1(4)(8)^2 \end{array} \right] * 2 + \left[\frac{1(20)^3}{12} \right]$$



$$I_x = 4191.833 \text{ cm}^4$$

$$t = 1 \text{ cm}$$

$$q = \left(\frac{\Phi_y}{I_x} \right) * S_x$$

$$q_1 = (1)(2)(7) = 14$$

$$q_1 = 1(4)(8) \left(\frac{\Phi_y}{I_x} \right) = 32 \left(\frac{\Phi_y}{I_x} \right)$$

$$\Delta = 2 \left(\frac{\Phi_y}{I_x} \right)$$

$$q_2 = [32 + (1)(10)(10)] \left(\frac{\Phi_y}{I_x} \right) = 132 \left(\frac{\Phi_y}{I_x} \right)$$

$$q_3 = (1)(5)(10) \left(\frac{\Phi_y}{I_x} \right) = 50 \left(\frac{\Phi_y}{I_x} \right) \quad (5)$$

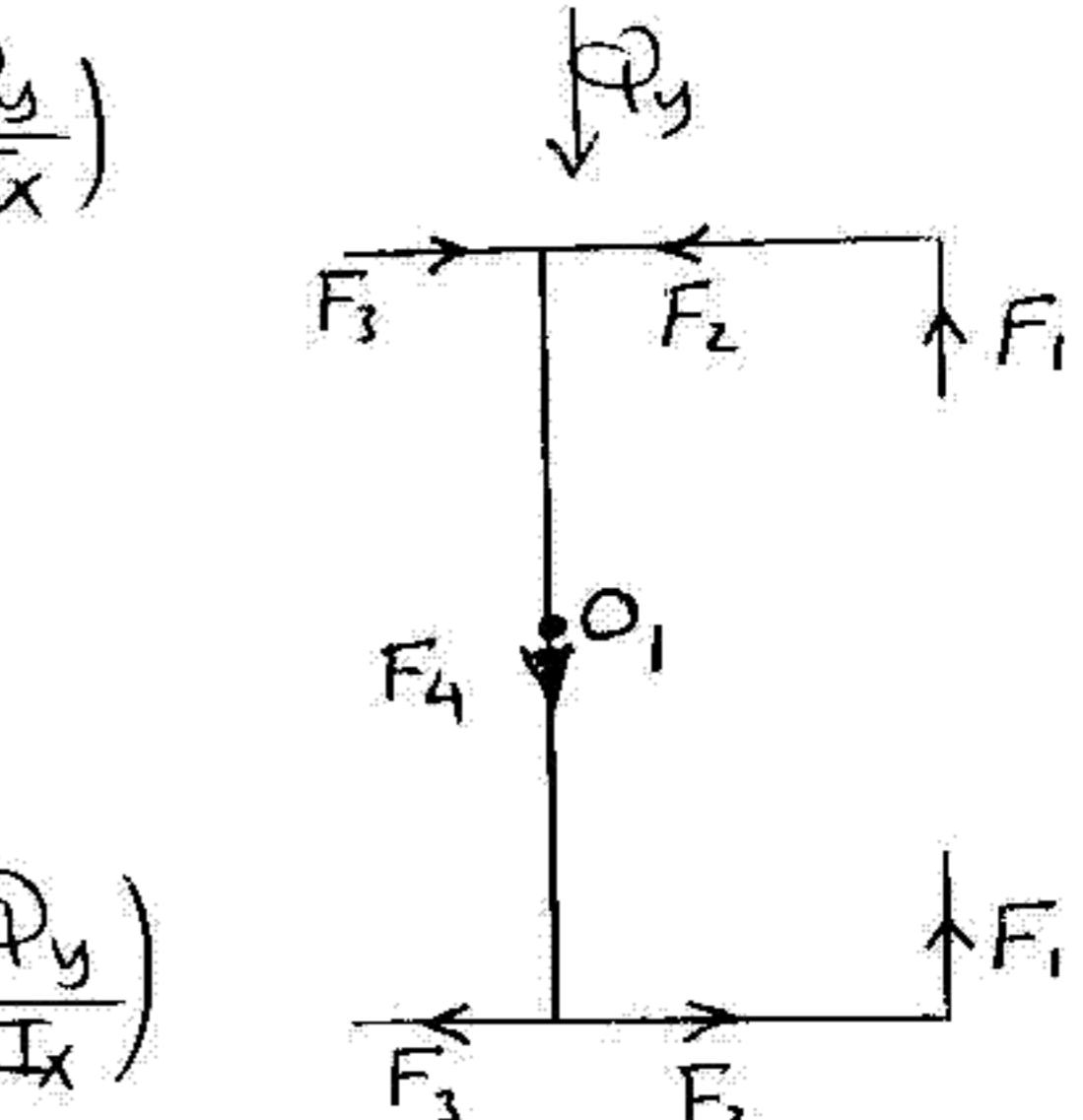
$$q_4 = q_2 + q_3 = 182 \left(\frac{\Phi_y}{I_x} \right)$$

$$q_5 = [182 + (1)(10)(5)] \left(\frac{\Phi_y}{I_x} \right) = 232 \left(\frac{\Phi_y}{I_x} \right)$$

$$F_1 = \left(\frac{1}{2} \right) q_1 (4) = 64 \left(\frac{\Phi_y}{I_x} \right) - \frac{2}{3} \left(2 \frac{\Phi_y}{I_x} \right) (4) = 58.667 \left(\frac{\Phi_y}{I_x} \right)$$

$$F_2 = \left(\frac{q_1 + q_2}{2} \right) (10) = 820 \left(\frac{\Phi_y}{I_x} \right) \quad (5)$$

$$F_3 = q_3 * \frac{1}{2} * 5 = 125 \left(\frac{\Phi_y}{I_x} \right)$$



$$e = 3.894 \text{ cm} \quad (5)$$

$$F_4 = \left[(182 * 20) + \left(\frac{2}{3} * 50 * 20 \right) \right] \frac{\Phi_y}{I_x} = 4306.667 \left(\frac{\Phi_y}{I_x} \right)$$

8/10

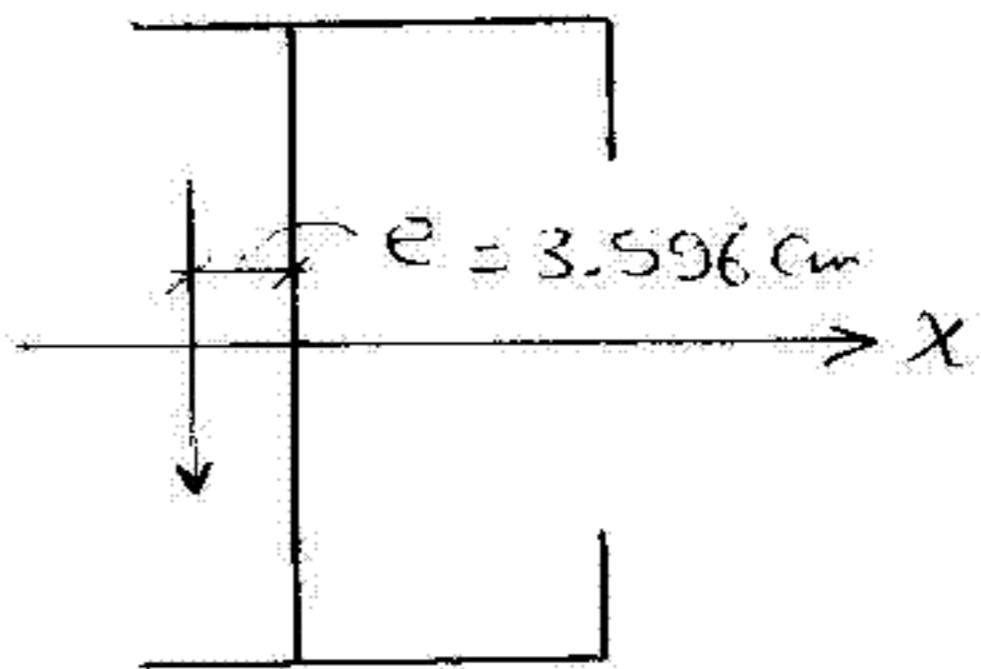
$$\Sigma M_{\omega_1 \infty}$$

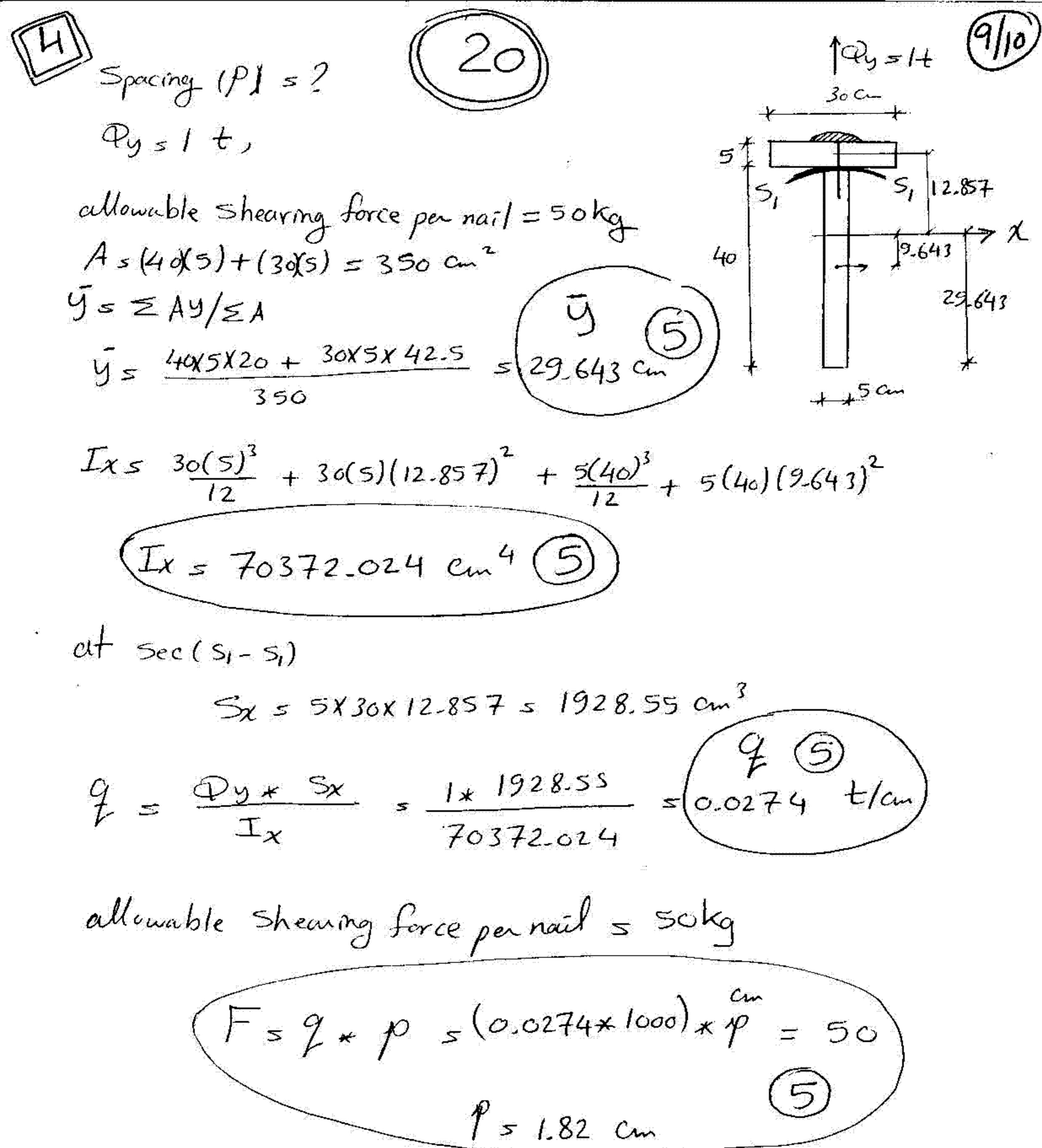
$$2 * F_1 * 10 + 2 * F_2 * 10 + 0 - F_3 * 2 * 10 = \Phi_y * e$$

$$2(58.67) \frac{\Phi_y}{I_x} * 10 + 2(820) \frac{\Phi_y}{I_x} * 10 - 2(125) \frac{\Phi_y}{I_x} * 10$$

$$= \Phi_y * e$$

$$e = 3596 \text{ cm}$$





5

Vertical deflection at point (d)

10/10

20

$$EI (\text{part abcd}) = 50 \ 000 \text{ m}^2 \cdot t$$

$$EA (\text{tie}) = 40 \ 000 \text{ t}$$

we will draw the B.M.D.

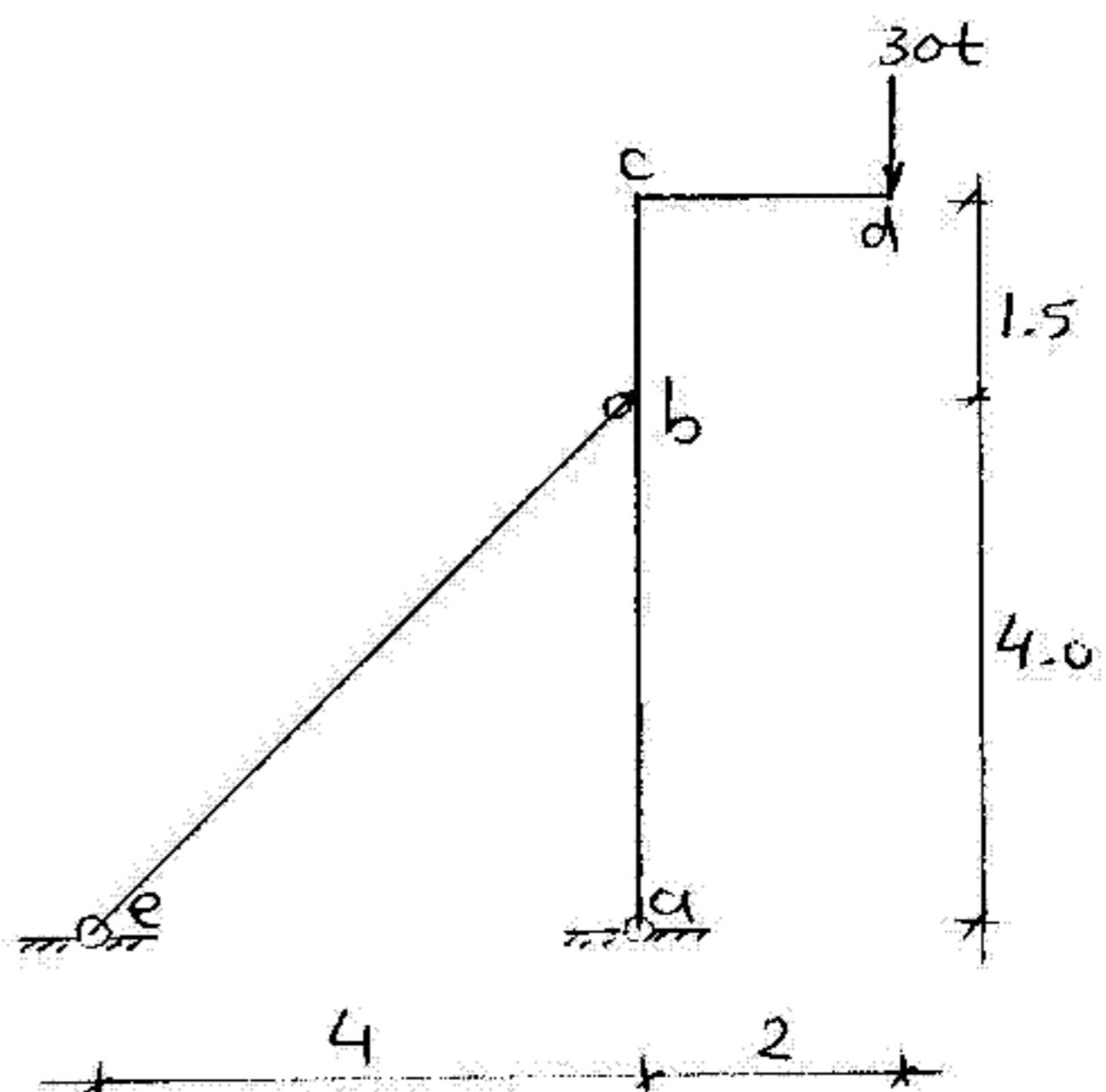
$$\text{the force in the tie} = 15\sqrt{2} + (\text{ten.})$$

$$y_d = \int \frac{M_0 M_1}{EI} dl + \int \frac{N_0 N_1}{EA} dl$$

$$= \frac{1}{EI} \left[\frac{2 \times 60 \times 2}{3} + 60(1.5)(2) + \frac{60 \times 2 \times 4}{3} \right]$$

$$+ \frac{1}{EA} \left[\frac{1}{\sqrt{2}} \times 15\sqrt{2} \times 4\sqrt{2} \right]$$

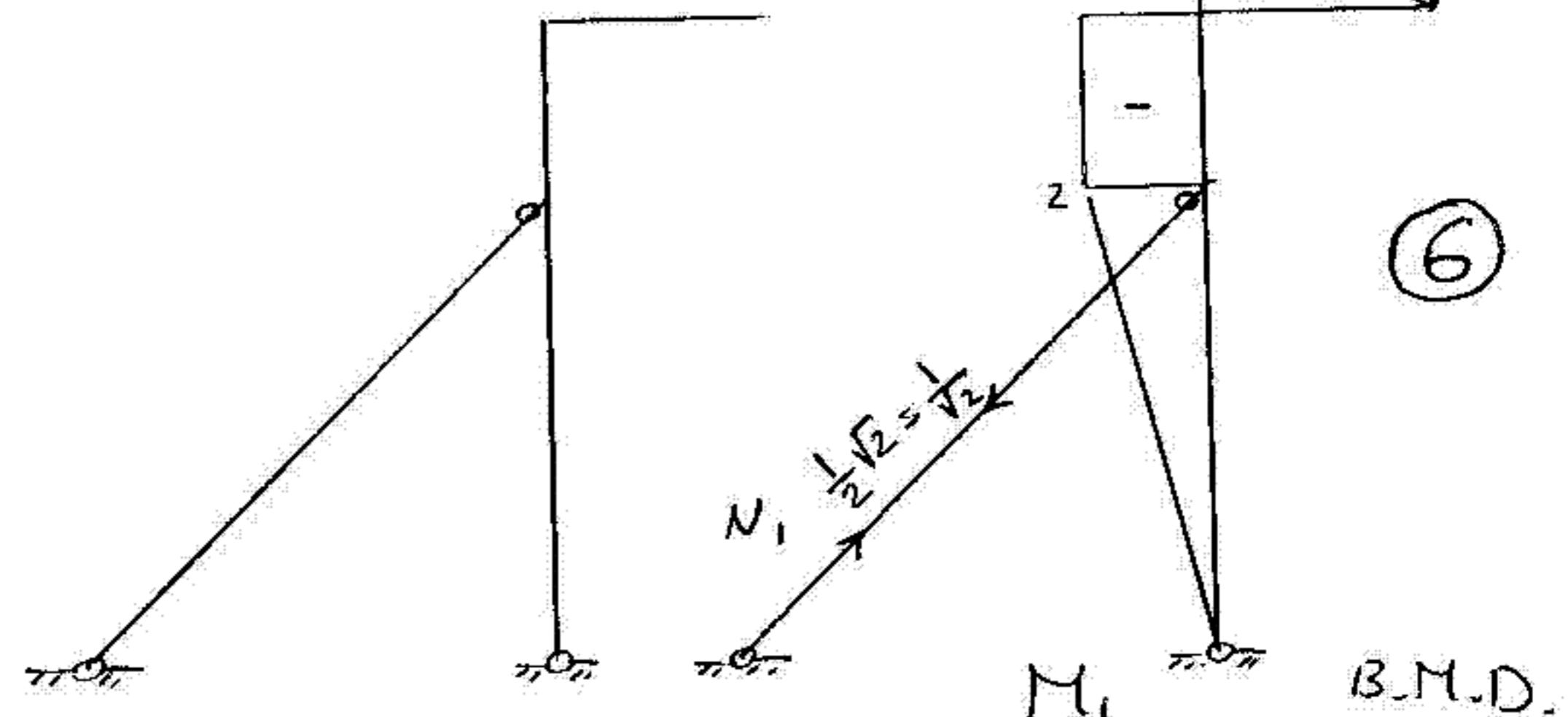
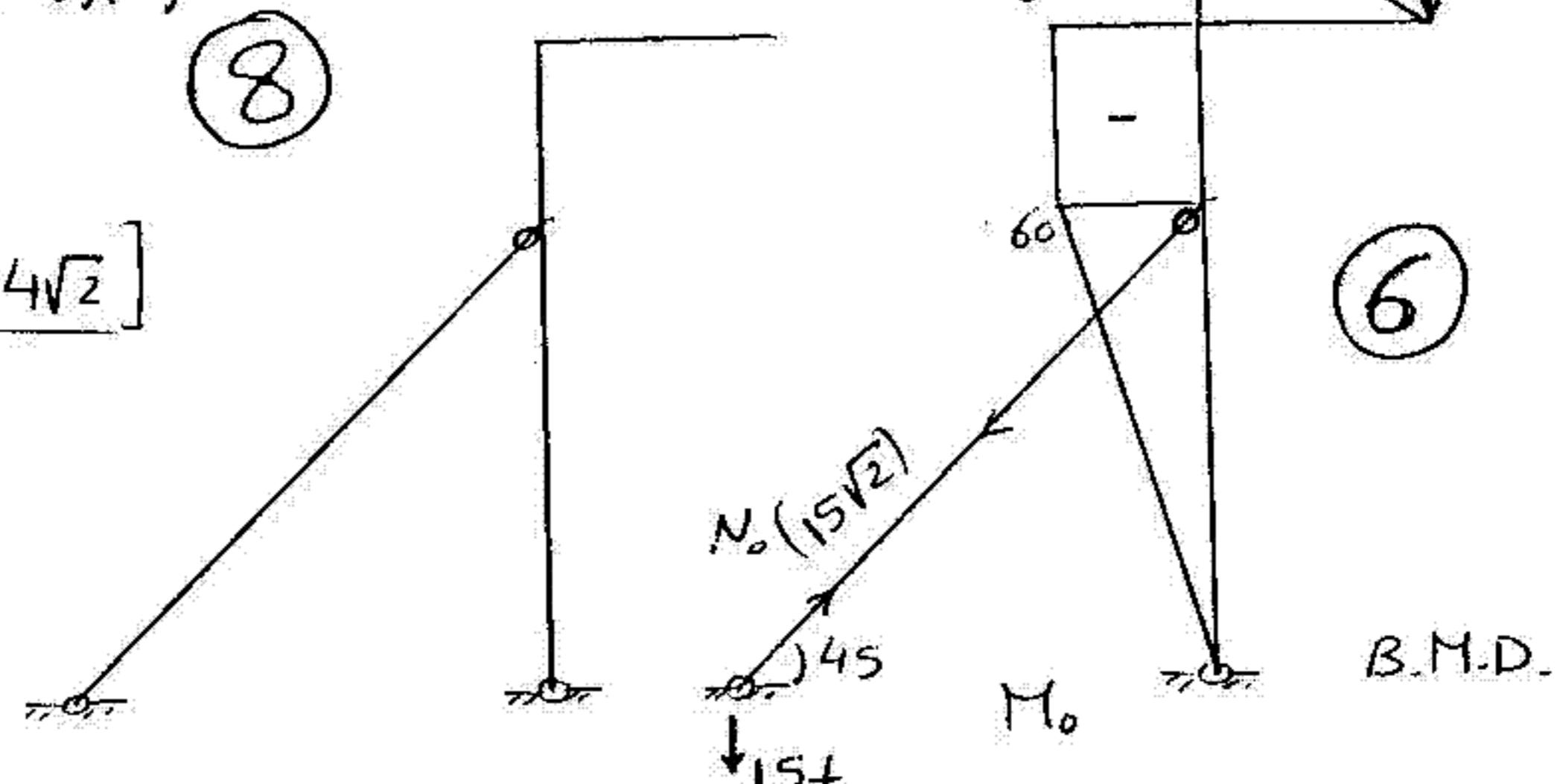
⑧

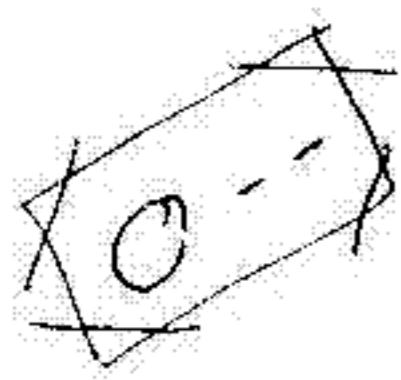


$$y_d = 0.0105 \text{ m} \downarrow$$

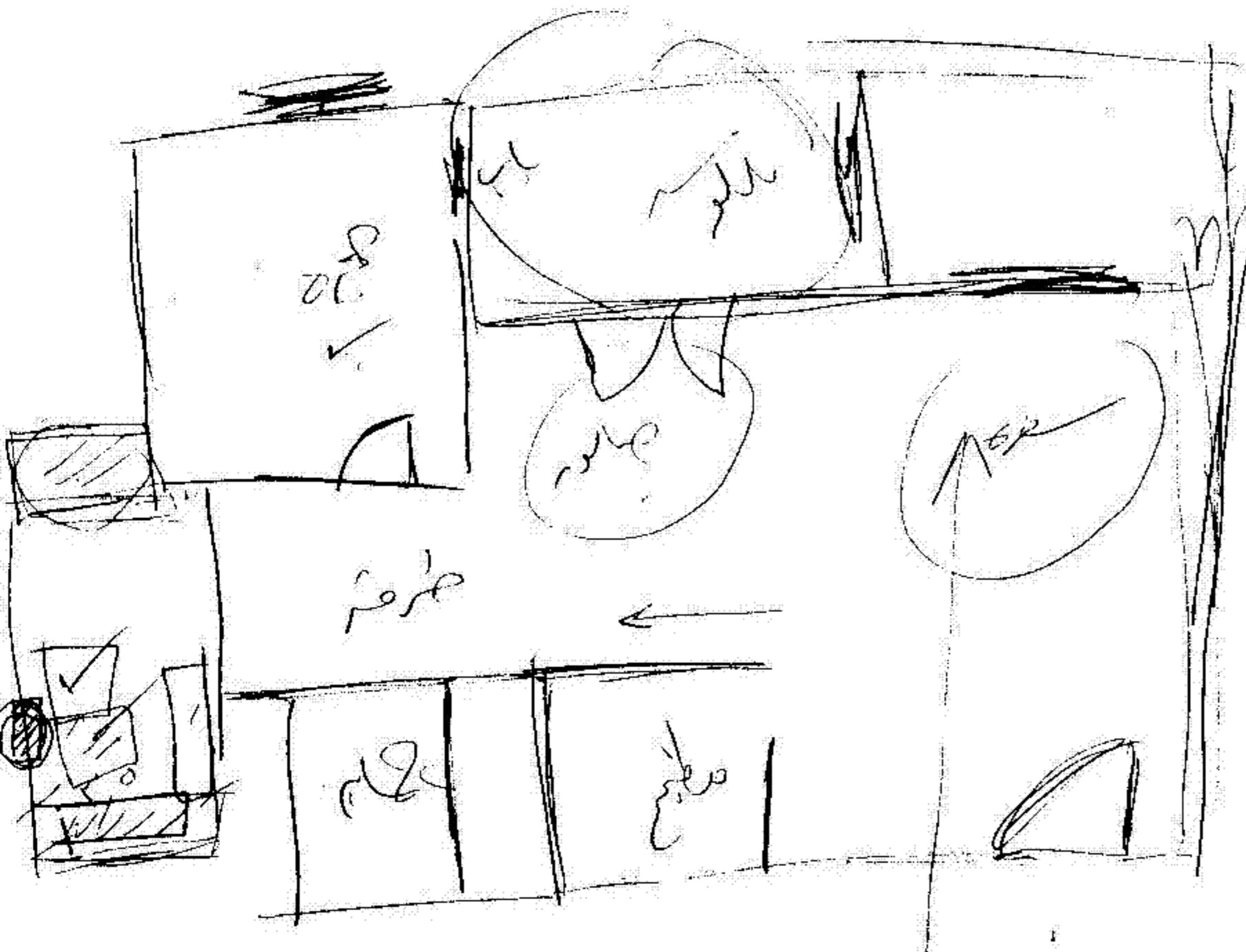
\downarrow

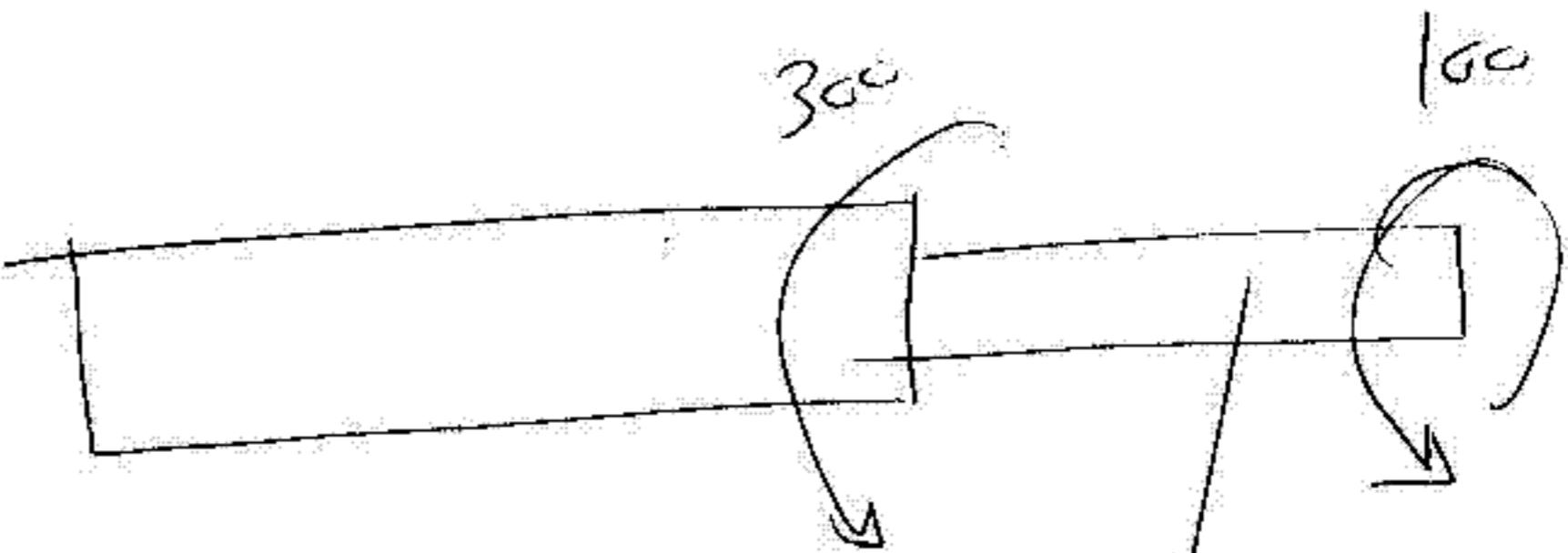
$$= 1.05 \text{ cm} \downarrow$$





ext, ext





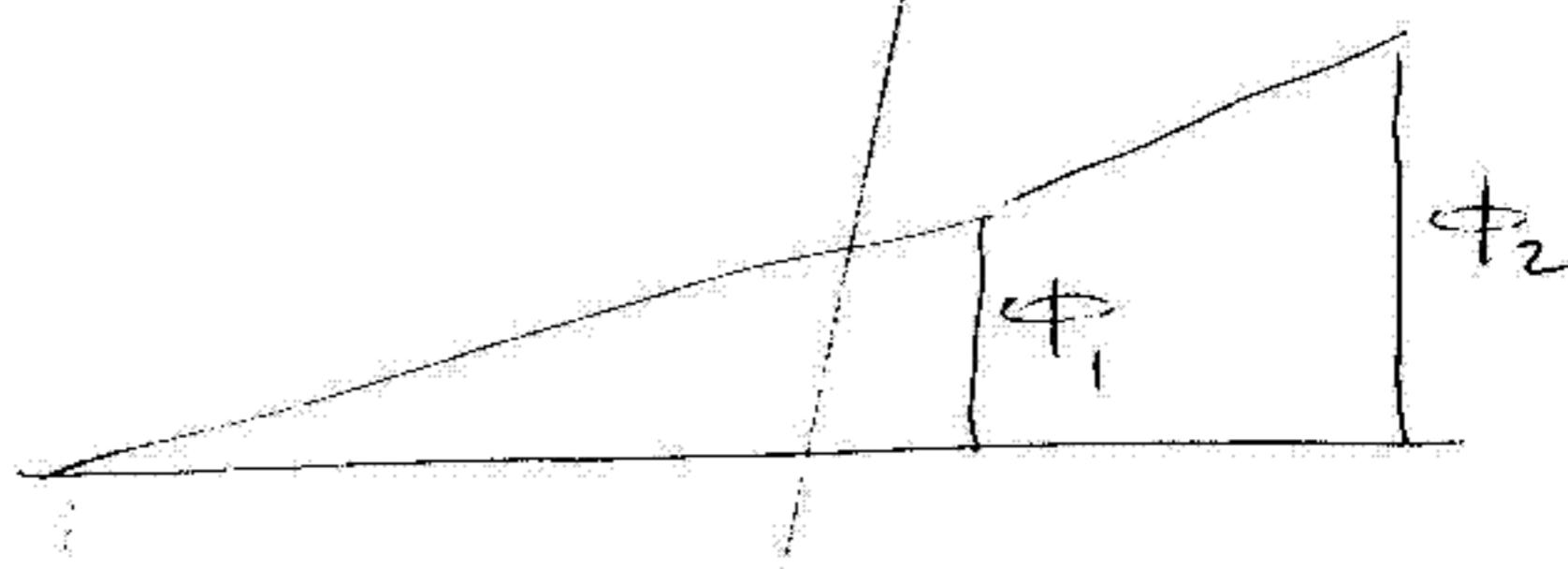
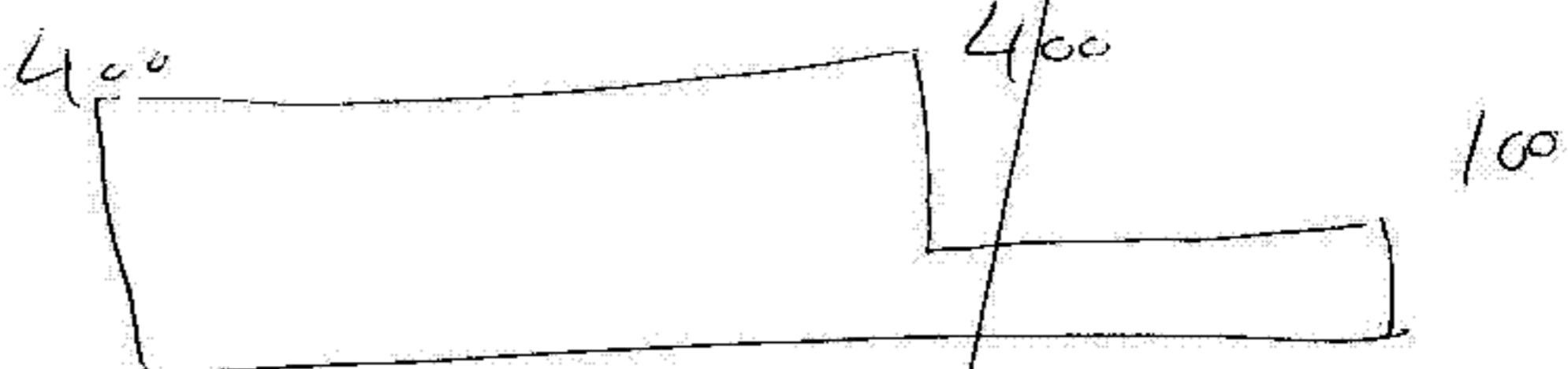
Tall's Soekglaze



$d_s?$

$\phi_s?$

$G = 500 \text{ kg/cm}^2$

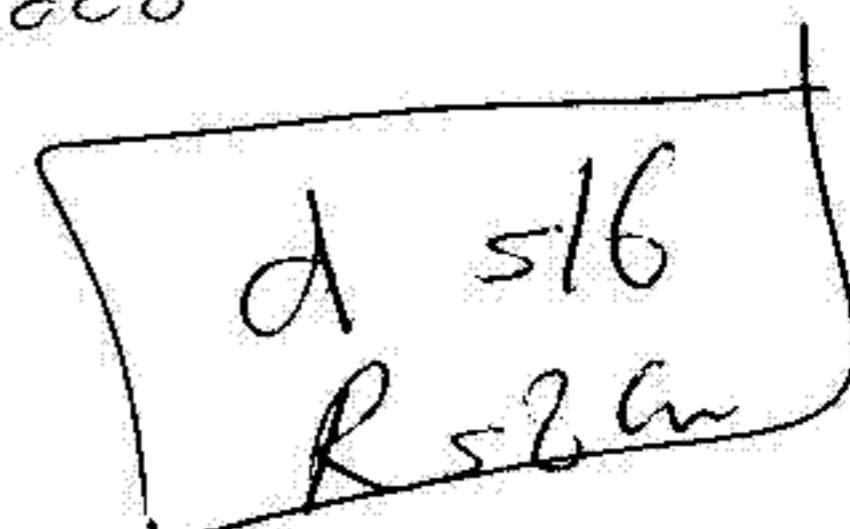


$$T_s = \frac{M_t R}{I_p} = \frac{\frac{t_m}{d} \cdot \frac{d_m}{d}}{\pi d^4 / 32} \leq \frac{500 \times 10^{-3}}{10^{-4}}$$

$$\frac{2037.18}{100d^3} \leq 5000$$

$$d^3 \geq 0.407$$

$$d \geq 0.74 \text{ m}$$



$\phi_s \phi_1 + \phi_2$