



- Start every question in a new page.

(1) (a) A vertical concentric annulus, with outer radius ( $r_o$ ) and inner radius ( $r_i$ ), is lowered into fluid of surface tension ( $\sigma$ ) and contact angle ( $\theta < 90^\circ$ ). Derive an expression for the capillary rise ( $h$ ) in the annular gap, if the gap is very narrow.

$\sum F_y = 0$

Figure 1

$$\sigma \cos \theta (2\pi r_o + 2\pi r_i) = \gamma \pi (r_o^2 - r_i^2) h$$

$$\therefore h = \frac{2\sigma \cos \theta}{\gamma (r_o - r_i)}$$

(b) A layer of water flows down an inclined fixed surface with the velocity profile shown in figure 2. Determine the magnitude and direction of the shearing stress that the water exerts on the fixed surface for  $U = 2$  m/s and  $h = 0.1$  m. ( $\mu = 1.12 \times 10^{-3}$ )

$\tau = \mu \frac{du}{dy}$

$\frac{du}{dy} = U \left( \frac{2}{h} - \frac{2y}{h^2} \right)$

at  $y = 0$   $\frac{du}{dy} \Big|_{y=0} = \frac{2U}{h}$

$\tau = \mu \left( \frac{2U}{h} \right) = 1.12 \times 10^{-3} \times 2 \times \frac{2}{0.1}$

$= 4.48 \times 10^{-2} \text{ N/m}^2$  in the flow direction

$\frac{u}{U} = \frac{2y}{h} - \frac{y^2}{h^2}$

$u = U \left( \frac{2y}{h} - \frac{y^2}{h^2} \right)$

(2) (a) A Circular gate ABC (figure 3) is hinged at B. Compute the force P just sufficient to keep the gate from opening when  $h = 8.0$  m. Neglect atmospheric pressure.

$F = \gamma A \bar{h} = 9810 \times \pi \times 2^2 \times 8 = 8\pi t = 25.13t$

$= 246552 \text{ N} = 25.13 \text{ t}$

$Z = \frac{I_x}{A \bar{y}} = \frac{\frac{\pi (1)^4}{4}}{8\pi} = 0.03125 \text{ m}$

$\sum M_B = 0 \quad P \times 1.0 = 246552 \times 0.03125$

$\therefore P = 7704.76 \text{ N} = 0.7856 \text{ t}$

(b) Analyze problem (2 - a) for arbitrary depth ( $h$ ) and gate radius ( $R$ ) and derive a formula for the opening force ( $P$ ). Prove that the force  $P$  is not dependent on the depth  $h$ .

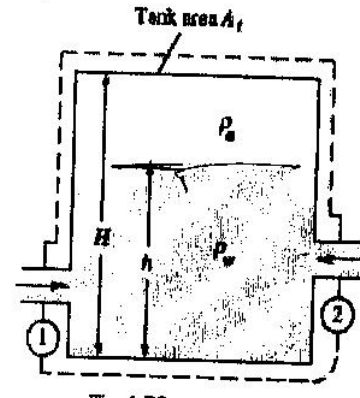
$F = \gamma A \bar{h} = \gamma \pi R^2 h$

$Z = -\frac{I_x}{A \bar{y}} = -\frac{R^2}{4h}$

$\sum M_B = 0 \Rightarrow 0 = (\gamma \pi R^2 h) \times \frac{R^2}{4h} = PR$

$\therefore P = \frac{\gamma \pi R^3}{4}$  independent on

- (b) The tank in Figure 7 is being filled with water by two inlets. The area of the tank is  $A_t$ . The water height is  $(h)$ . Compute  $(dh/dt)$  for  $D_1 = 5.0$  cm,  $D_2 = 7.0$  cm,  $V_1 = 1.0$  m/s and  $V_2 = 0.7$  m/s and  $A_t = 0.2$  m<sup>2</sup>.



$$\begin{array}{r} 140.95 \\ 118.45 \\ \hline 22.50 \end{array}$$

Figure 7

$$\begin{array}{r} 4.0 \\ 1.0 \\ \hline 5.0 \\ 1.0 \\ \hline 6.0 \\ 1.0 \\ \hline 7.0 \end{array}$$

- (c) A closed rectangular tank 1.2 m high, 2.5 m long, and 1.5 m wide is filled with water. The pressure at the top is raised to 60 kN/m<sup>2</sup>. Calculate the pressure at the corners of this tank when it is decelerated horizontally along the direction of its length at 5.0 m/s<sup>2</sup>. Calculate the force on the ends of the tank, and check their difference by Newton's law.

$$\tan \theta = \frac{a}{g} = \frac{h}{2.5} \quad \text{Best Wishes}$$

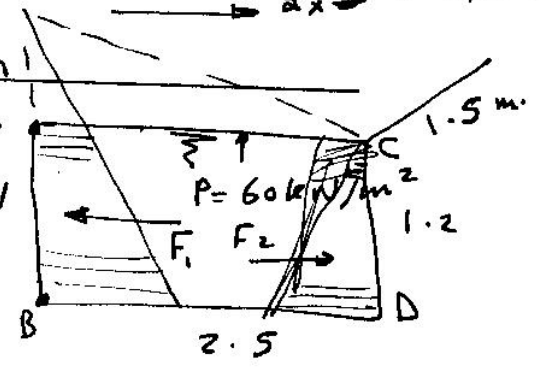
$$\frac{5}{9.81} = \frac{h}{2.5} \rightarrow h = 1.275 \text{ m}$$

Examiners:  
Dr. Ayman Georges  
Dr. Nabil A. Awadallah

$$\begin{aligned} P_A &= 72.495 \text{ kN} \\ P_B &= 84.255 \text{ kN} \\ P_C &= 60 \text{ kN/m}^2 \\ P_D &= 71.76 \text{ kN} \end{aligned}$$

$$\begin{aligned} P_A &= \gamma h + 60 = 72.495 \text{ kN/m}^2 \\ P_B &= \gamma (h + 1.2) + 60 = 84.255 \text{ kN/m}^2 \\ P_C &= 60 \text{ kN/m}^2 \\ P_D &= \gamma (1.2) + 60 = 71.76 \text{ kN/m}^2 \end{aligned}$$

$a = 5 \text{ m/s}^2$



$$\begin{aligned} F_1 &= F_{AB} = \frac{(P_A + P_B)}{2} \times 1.2 \times 1.5 = 140.951 \text{ kN} \\ F_2 &= \frac{(P_C + P_D)}{2} \times 1.2 \times 1.5 = 118.4517 \text{ kN} \\ \Delta F &= F_1 - F_2 = 22.5 \text{ kN} \end{aligned}$$

$$\Sigma F_x = m \times a = \frac{\gamma \times 2.5 \times 1.2 \times 1.5}{9.81} \times 5 = 22.5 \text{ kN}$$

From (1) & (2)  $\Sigma F_x = m \times a = \Delta F \quad \text{ok}$

۱. د. محمد حسن  
 مکتبہ  
 کراچی

Model Answer

① (a)  $\Sigma F_y = 0.0 \quad \bar{F}_0 \uparrow = w \downarrow$

$\sigma \cos \theta (2\pi r_0 + 2\pi r_1) = \sigma (\pi r_0^2 - \pi r_1^2) h$

$\therefore h = \frac{2\sigma \cos \theta}{\sigma (r_0 - r_1)}$

(b)  $\tau = \mu \frac{du}{dy}$  &  $u = U \left( \frac{2y}{h} - \frac{y^2}{h^2} \right) \Rightarrow \frac{du}{dy} = U \left( \frac{2}{h} - \frac{2y}{h^2} \right)$   
 $\frac{u}{y} \Big|_{y=0} = \frac{2U}{h} = \frac{2 \times 2}{0.1} = 40 \text{ Vsec.}$

$\therefore \tau = 40 \mu = 40 \times 1.12 \times 10^{-3} = 4.48 \times 10^{-2} \text{ N/m}^2 \text{ inflow dir.}$

② (a)  $F = \sigma A \bar{h} = 9810 \times \pi (1)^2 \times 8 = 246552.2 \text{ N} = 25.13 \text{ t} = 8\pi \text{ t}$

$z = \frac{I_{c.g.}}{Ay} = \frac{\pi (1)^4 / 4}{\pi (1)^2 \times 8} + 8 = 8.03125 \text{ m.}$

$z = 0.03125 \text{ m}$

$\Sigma M \text{ about } B = 0.0 \quad F + z = P \times r$

$F + 0.03125 = P \times 1 \Rightarrow P = 7704.76 \text{ N} = 0.785 \text{ t}$

(b)  $F = \sigma A \bar{h} = \sigma \pi R^2 \times h = \sigma \pi R^2 h$

$z = \frac{I_{c.g.}}{Ay} = \frac{\pi R^4 / 4}{\pi R^2 \times h} = \frac{R^2}{4h}$

$\Sigma M \text{ about } B = 0.0 \therefore P \times R = \sigma \pi R^2 h + \frac{R^2}{4h}$

$P = \frac{\sigma \pi R^3}{4}$

③  $\tan \theta = \frac{2x}{g} = \frac{h}{2.5} \Rightarrow h = 1.275 \text{ m}$

$F_1 = P_A = \frac{(P_A + P_B)}{2} \times 1.2 \times 1.5 = 140.951 \text{ kN}$

$F_2 = P_D = \frac{(P_C + P_D)}{2} \times 1.2 \times 1.5 = 118.4517 \text{ kN}$

$\Delta F = F_1 - F_2 = 22.5 \text{ kN} \rightarrow \textcircled{1}$

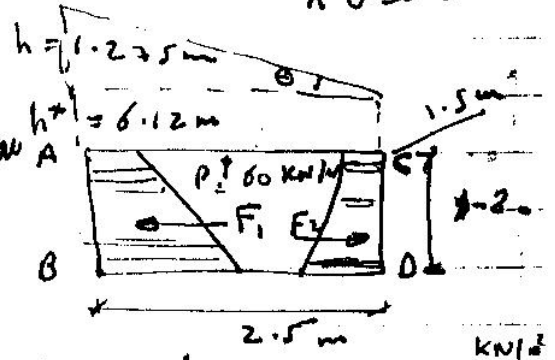
$\Sigma F_x = \text{mass} \times \text{acc.} = \frac{\sigma}{g} \times 2.5 \times 1.2 \times 1.5 \times 5$

$= \frac{9.81}{9.81} \times 2.5 \times 1.2 \times 1.5 \times 5 = 22.5 \text{ kN}$

$\rightarrow \textcircled{2}$

From ① & ②

$\Sigma F_x = \text{mass} \times \text{acc.} = \Delta F \quad \times$



$P_A = \sigma h + 60 = 72.495$

$P_B = \sigma (h + 1.2) + 60 = 84.255$

$P_C = 60 \text{ kN/m}^2$

$P_D = \sigma (1.2) + 60 = 71.76 \text{ kN/m}^2$

Question 3 (b)

$$F_{h_1} = (1.0) (2) (4 \times 8) = 64 \text{ t}$$

$$F_{h_2} = (1) (2 \times 8) = 16 \text{ t}$$

$$F_{v_1} = \frac{\pi}{2} (4) (8) = 16\pi = 50.27 \text{ t}$$

$$F_{v_2} = \frac{\pi}{4} (4) (8) = 8\pi = 25.13 \text{ t}$$

$$R_H = 64 - 16 = 48 \text{ t}$$

$$R_V = (0.8) 4\pi (8) - 8\pi - 16\pi = 1.6\pi = 5.03 \text{ t}$$

Question 4 (b)

$$MB = \frac{I}{y} = \frac{(1) (4.6)^3 / 12}{(1.2) (4.6)} = 1.47 \text{ m}$$

$$BG = 0.45 + 1.2 = 1.65 \text{ m}$$

Since  $BG < MB \rightarrow$  unstable

Question 5 (a)

$$0.6 \gamma_w \frac{V}{d} = \gamma_{\text{Fluid}} (0.75 \frac{V}{d}) \Rightarrow \gamma_{\text{Fluid}} = 7832 \text{ N/m}^3$$

$$\begin{aligned} \text{Air Pressure} &\Rightarrow 0 - (7832) (0.4) = \text{Pa}_{\text{air}} \\ &\Rightarrow -3130 \text{ (Pa gauge)} \text{ [N/m}^2\text{]} \end{aligned}$$

Question 5 (b)

$$\frac{dh}{dt} = \frac{A_1 V_1 + A_2 V_2}{A_t} = \frac{Q_1 + Q_2}{A_t}$$

$$Q_1 = 1.963 \text{ l/s}, \quad Q_2 = 2.694 \text{ l/s}$$

$$\frac{dh}{dt} = \frac{4.657}{(0.2) (1000)} = 0.0233 \text{ m/s} = 2.33 \text{ cm/s}$$